Department of Automotive Technologies – Vehicle Mechanics Fundamentals



Gábor Sipos

Lecture 4

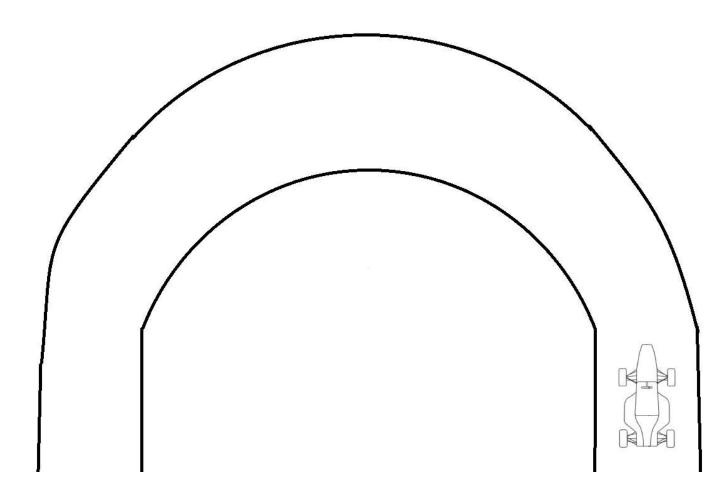
03.07.2022.

Basic information

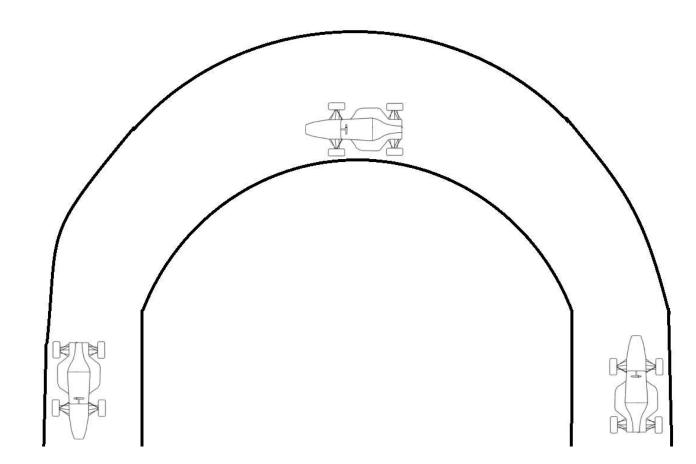


Week nr.	Date	Lecture (Monday)		Lab (date+1;Tuesday)	
1	12th Feb	1	General information, Tyre, Driving force	1	Lab
2	19th Feb	2	Longitudinal and lateral behaviour		
3	26th Feb	3	Concepts and over/understeer	2	Lab
4	4th Mar	4	Weight transfer		
5	11th Mar	5	Bicycle model	3	Lab
6	18th Mar	T1	Midterm exam I. ONLINE		
7	25th Mar	6	Braking and brakes ONLINE	4	Lab ONLINE
8	1st Apr	-	Break		
9	8th Apr	7	Systems of the vehicle	T1 R	Exam 1 - subsequent
10	15th Apr	8	Quarter vehicle model ONLINE		
11	22th Apr		Break		Break
12	29th Apr	T2	Midterm exam II. ONLINE		
13	6th May	9	Tyre management	T2 R	Exam 2 - subsequent
14	13th May	10	Racecar engineering		























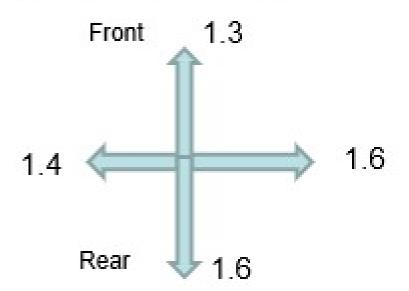




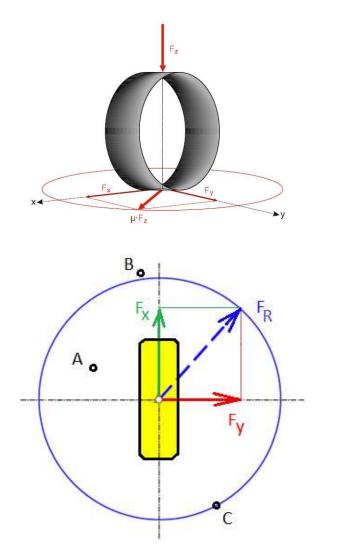




Different Friction Coefficient along Lateral and Longitudinal Axis





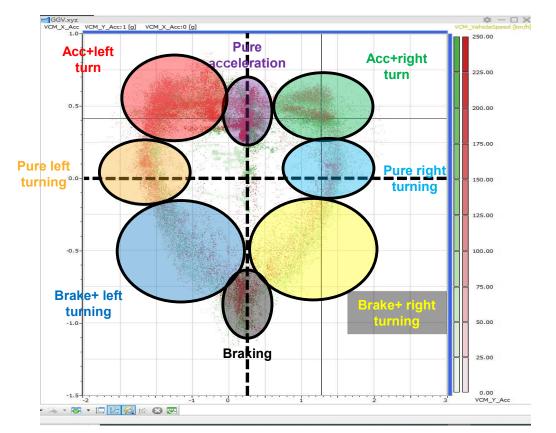


$$F_{\rm R} = \mu_{\rm max} F_{\rm z} \ge \sqrt{F_{\rm x}^2 + F_{\rm y}^2}$$

- F_x: longitudinal force; F_y: lateral force; F_z: normal force; F_R: resultant horizontal force;
- To which places can resultant horizontal force point?
 - A
 - B
 - C
- The tyre is able to accelerate and corner at the same time. How much of the different components can it use?



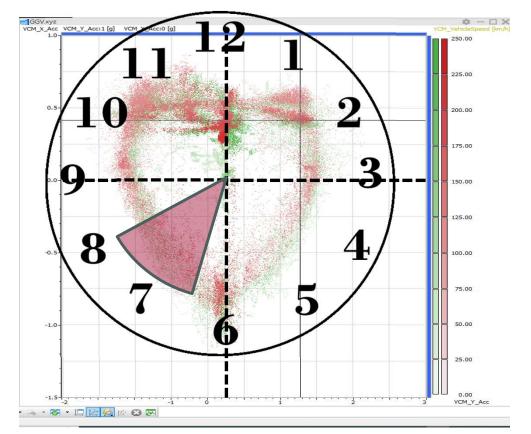




* Zero lat axis sensor calibration issue corr







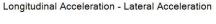
* Zero lat axis sensor calibration issue corr

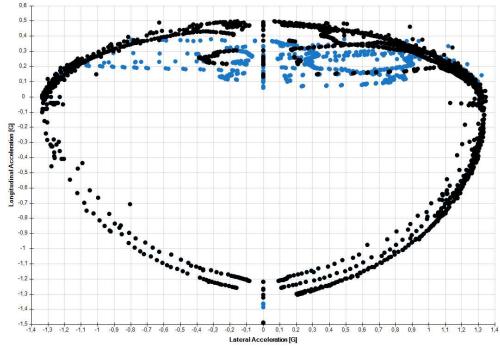




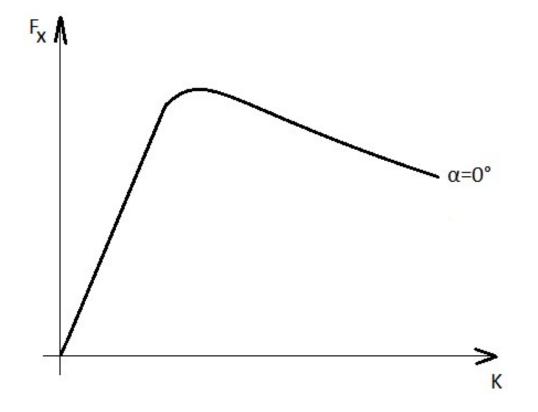
What could be the modification from black (base) to blue electric racecar?

- A changed tyre with more lateral performance
- B electric drivetrain changed to a lower power mode
- C electric drivetrain changed to a higher regenerative braking mode
- D suspension adjusted to reach higher lateral acceleration

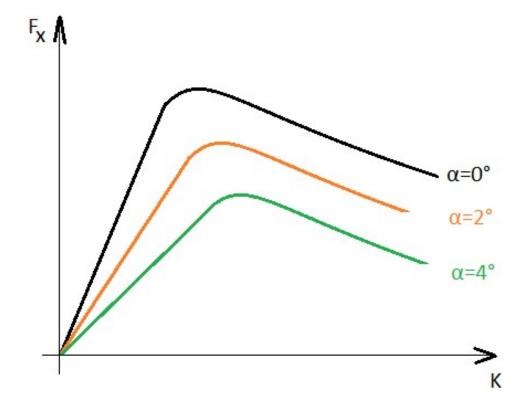




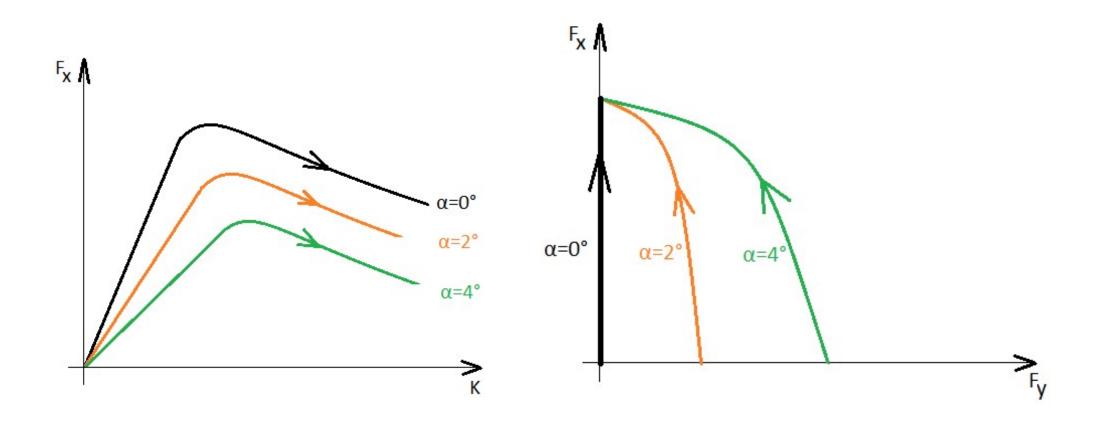




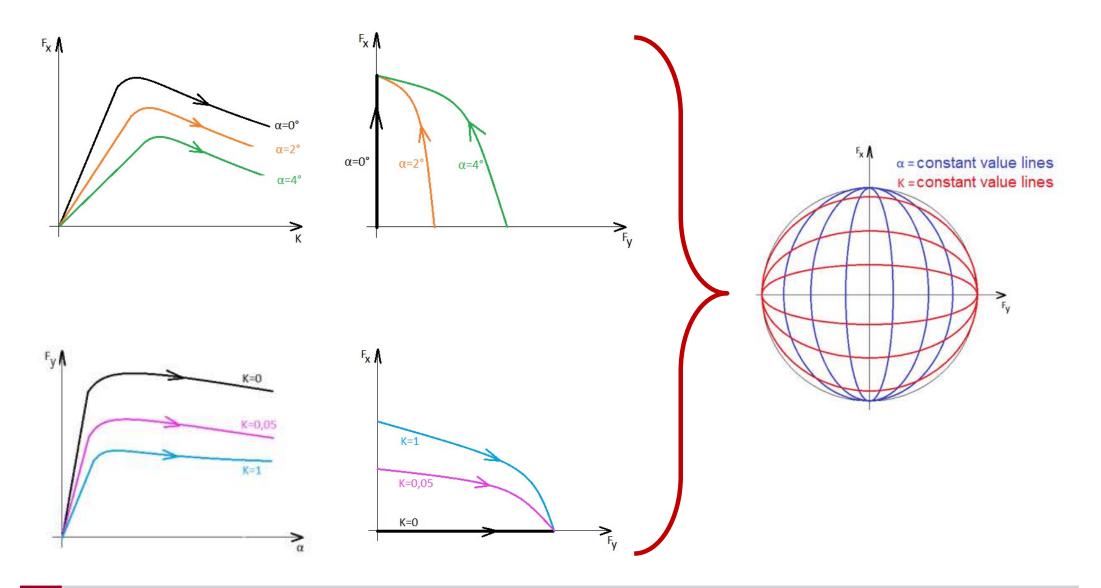




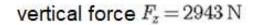


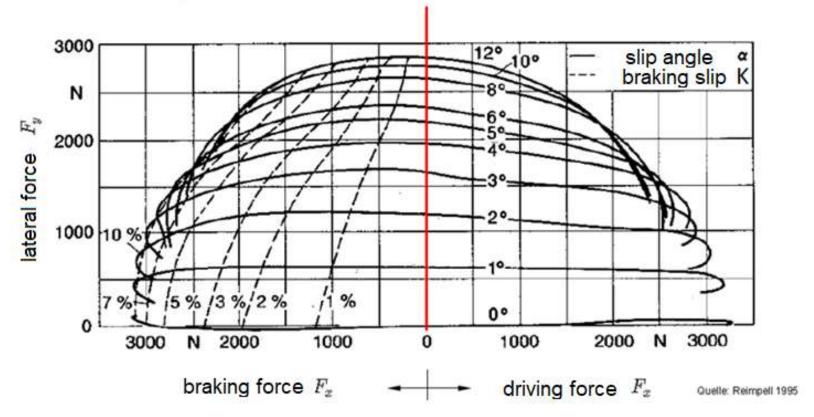






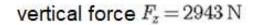


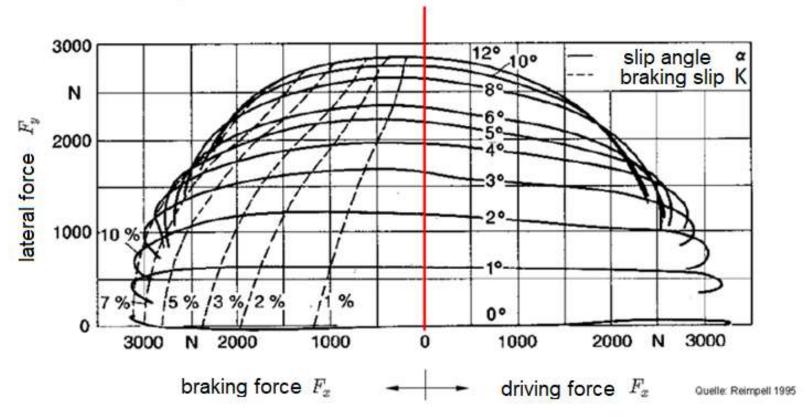




- What is the reason behind the elliptic shape of curves?
- Why the curves are asymmetrical to Fx=0 axis?



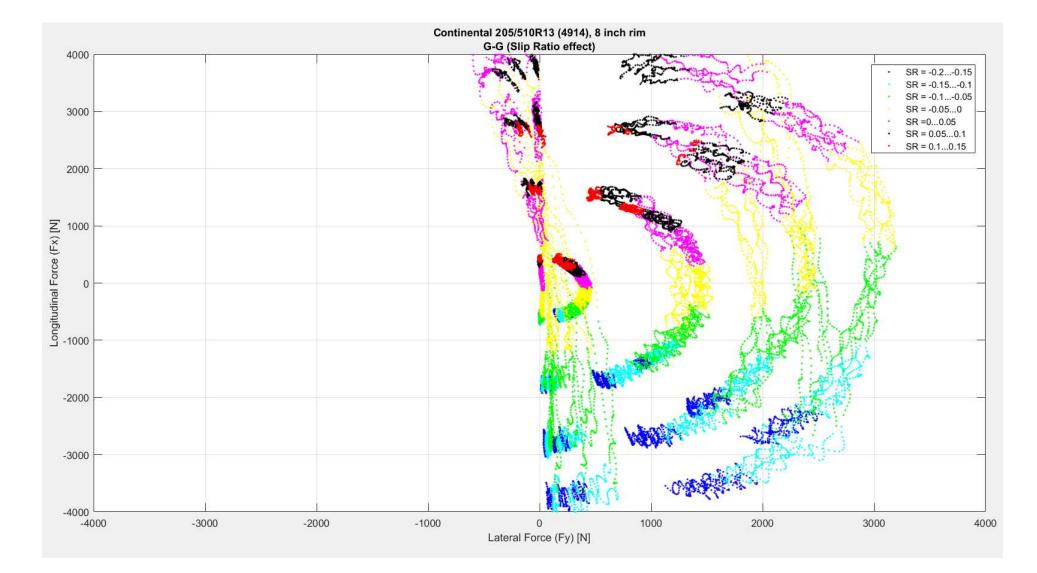




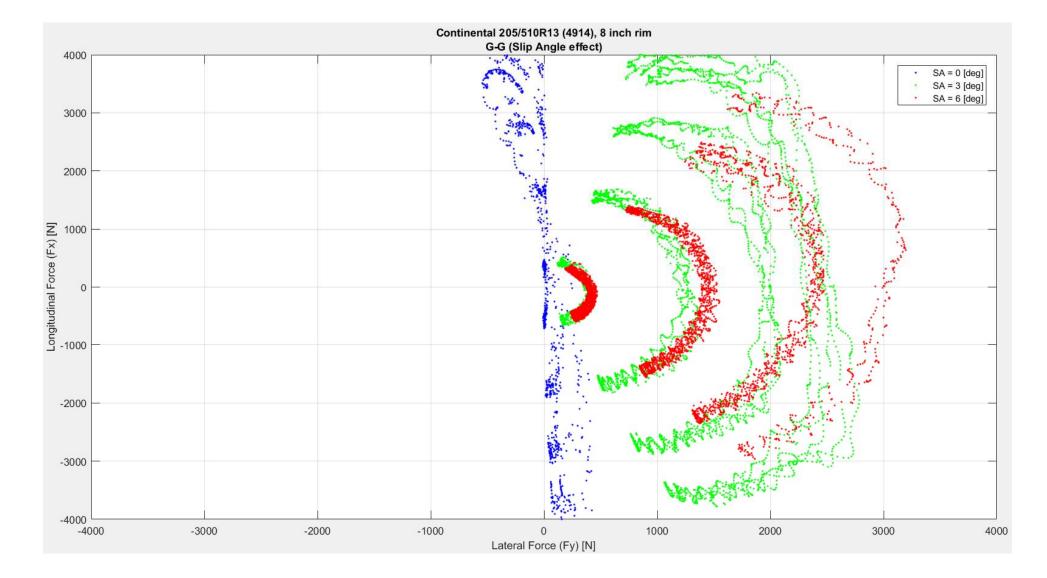
- What is the reason behind the elliptic shape of curves?
 Different values of friction coefficients to different directions.
- Why the curves are asymmetrical to Fx=0 axis?

Tyre behaviour is different in case of accelerating and braking.









Notation



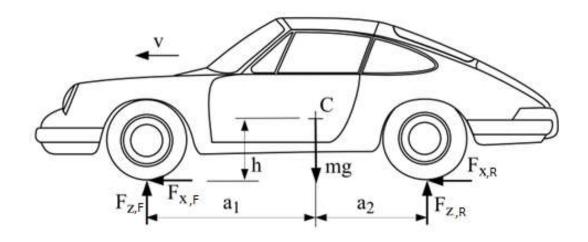


Dynam



	vertical dynamics	lateral dynamics	longitudinal dynamics
	$ \begin{array}{c} \hline m_{s} \\ \hline k_{s} \\ \hline m_{v} $	$\begin{array}{c} \delta \\ F_{yf} \\ \alpha_{f} \\ \beta \\ v \\ B \\ v \\ S \\ \alpha_{r} \\ \ell_{r} \\ M \\ \sigma \\ \rho_{M} \\ F_{yr} \end{array}$	$F_{z_{1}}F_{z_{1}}GF_{z_{1}}F_{z_{1}}F_{z_{1}}$
Features	 Vertical vibration Wheel loads comfort 	 Bicycle model steering Lateral acceleration Self aligning torque Critical speed 	 Acceleration and braking resistances: tyre,air, uphill – power requirement; Engine characteristics and gears Braking and driving forces
Parameters	m_s – structural weight m_w – wheel weight k_t – tyre stiffness k_s – spring stiffness k_d – damping u - movement	I – wheelbase I _f – CoG distance m – weight v – vehicle speed α – slip angle δ – steering angle	h – CoG height G – gravity force F _{xf} – front axle driving force F _{zf} – tyre forces at front





Static load of axles:

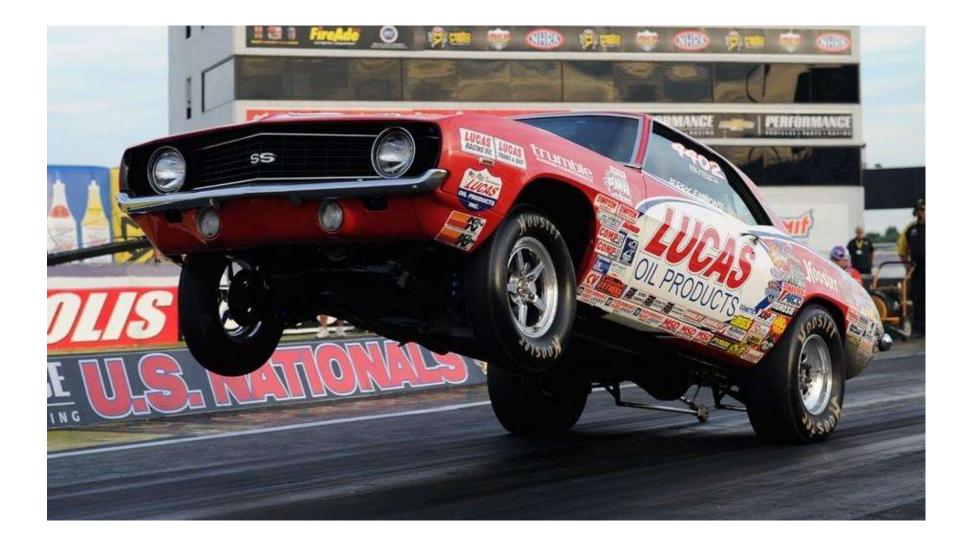
$$F_{z,F}^{st} = \mathbf{m} \cdot \mathbf{g} \cdot \frac{\mathbf{a}_2}{\mathbf{w}}$$
$$F_{z,R}^{st} = \mathbf{m} \cdot \mathbf{g} \cdot \frac{\mathbf{a}_1}{\mathbf{w}}$$

static load of tyres:

$$F_{z,1} = \frac{1}{2} \cdot F_{z,F}$$
$$F_{z,3} = \frac{1}{2} \cdot F_{z,R}$$

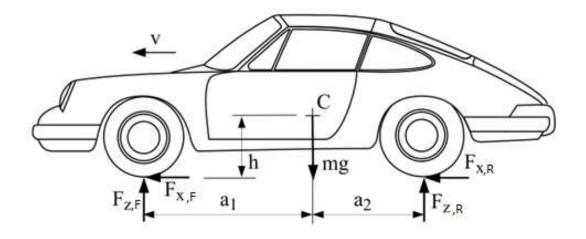


$$\dot{v} \neq 0$$



 $\dot{v}\neq 0$



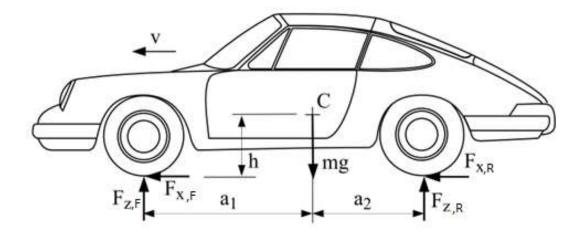


Law of motion:

- X direction: $m \cdot \dot{v} = F_{x,F} + F_{x,R}$
- Z direction: $0 = F_{z,F} + F_{z,R} m \cdot g$
- Moments: $0 = -F_{z,F} \cdot a_1 + F_{z,R} \cdot a_2 (F_{x,F} + F_{x,R}) \cdot h$

 $\dot{v} \neq 0$



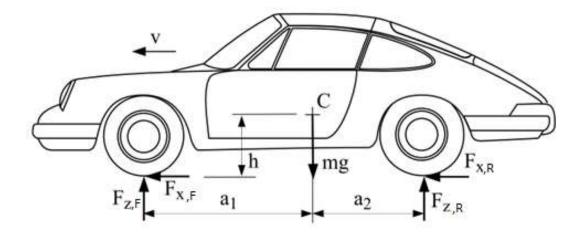


Dynamic load of axles:

 $F_{z,F} = F_{z,F}^{st} + F_{z,F}^{dyn} \longrightarrow$ $F_{z,R} = F_{z,R}^{st} + F_{z,R}^{dyn} \longrightarrow$

 $\dot{v}\neq 0$



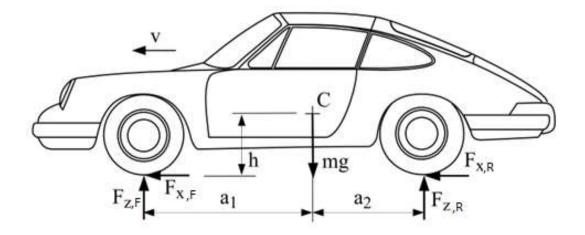


Dynamic load of axles:

$$F_{z,F} = F_{z,F}^{st} + F_{z,F}^{dyn} \longrightarrow F_{z,F}^{dyn} = -\mathbf{m} \cdot \dot{\mathbf{v}} \cdot \frac{\mathbf{h}}{\mathbf{w}}$$
$$F_{z,R} = F_{z,R}^{st} + F_{z,R}^{dyn} \longrightarrow F_{z,R}^{dyn} = \mathbf{m} \cdot \dot{\mathbf{v}} \cdot \frac{\mathbf{h}}{\mathbf{w}}$$

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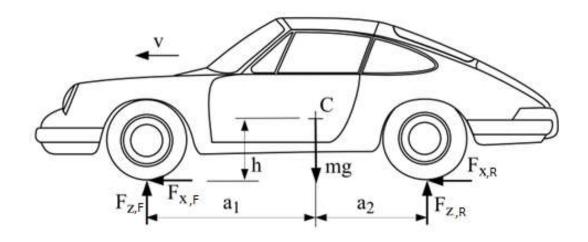
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Dynamic load of tyres:

$$F_{z,1} = \frac{1}{2} \cdot F_{z,F}$$
$$F_{z,3} = \frac{1}{2} \cdot F_{z,R}$$





Dynamic load of axles:

$$\begin{split} F_{z,F} &= m \cdot g \cdot \frac{a_2}{w} - \frac{h}{w} \cdot m \cdot \dot{v} \\ F_{z,R} &= m \cdot g \cdot \frac{a_1}{w} + \frac{h}{w} \cdot m \cdot \dot{v} \end{split}$$

Dynamic load of tyres:

$$F_{z,1} = \frac{1}{2} \cdot F_{z,F}$$
$$F_{z,3} = \frac{1}{2} \cdot F_{z,R}$$

What does weight transfer depend on?

- CoG height
- weight of car
- amount and direction of acceleration
- wheelbase





$$F_{z,R}^{dyn} = \mathbf{m} \cdot \dot{\mathbf{v}} \cdot \frac{\mathbf{h}}{\mathbf{w}}$$

What does weight transfer depend on?

- CoG height
- weight of car
- amount and direction of acceleration
- wheelbase

nothing else!

With increased wheelbase, the longitudinal weight transfer will be



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What does weight transfer depend on?

- CoG height
- weight of car
- amount and direction of acceleration
- wheelbase

nothing else!

With increased wheelbase, the longitudinal weight transfer will be

less

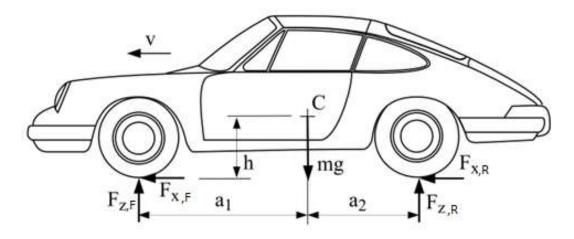


 $F_{z,R}^{dyn} = \mathbf{m} \cdot \dot{\mathbf{v}} \cdot \frac{\mathbf{h}}{\mathbf{w}}$



dynamic loads (and forces)

help me!

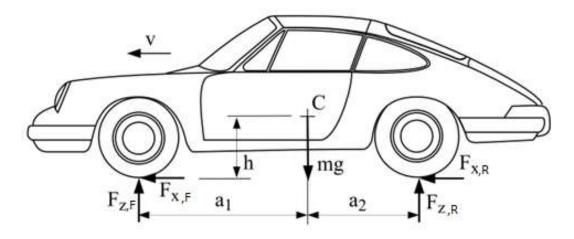


I'm going on a motorway with my own car, Waze warns of a pothole but it's too late, inevitable to go through, what should I choose?

- A. v should be 0, go through with constant speed
- *B.* \dot{v} should be negative, braking
- \mathcal{C} \dot{v} should be positive, accelerating
- D. none of the soultions above helps, damage will be the same



help me!



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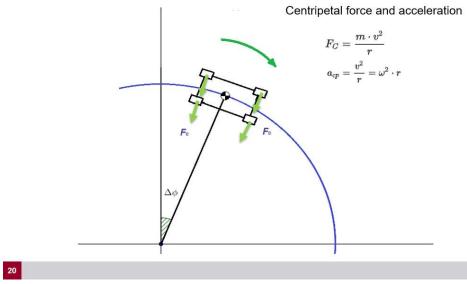
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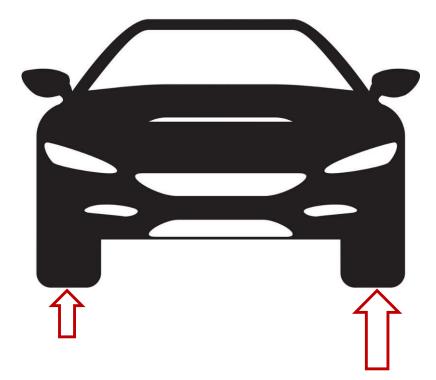




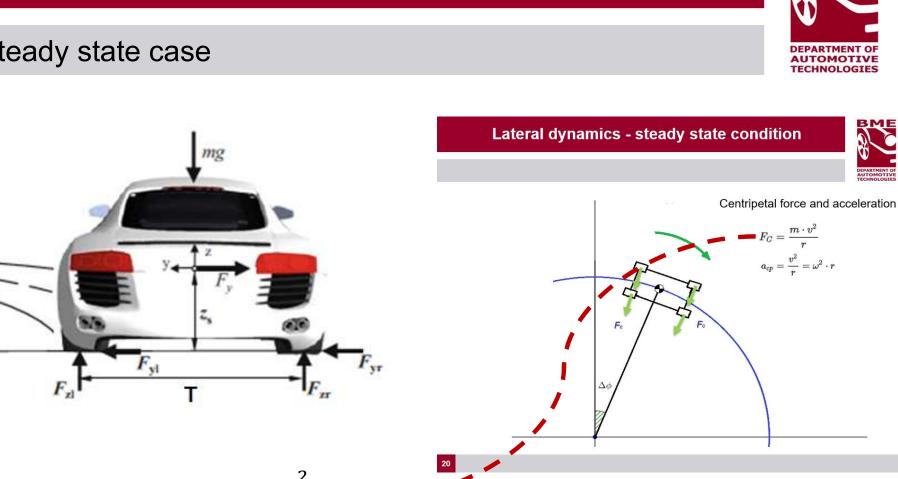
Lateral dynamics - steady state condition







Steady state case



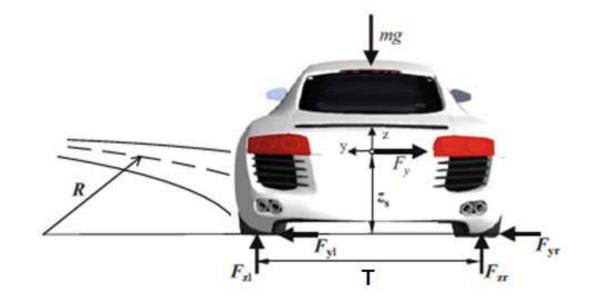
$$F_{cp} = \mathbf{m} \cdot a_y = m \cdot \frac{v^2}{R}$$



DEPARTMENT OF AUTOMOTIVE TECHNOLOGIES

Steady state case

$$F_{z,1} = \frac{1}{2} \cdot m \cdot g \cdot \frac{a_2}{w} - F_{cp} \cdot \frac{h}{T} \cdot \frac{a_2}{w}$$
$$F_{z,2} = \frac{1}{2} \cdot m \cdot g \cdot \frac{a_2}{w} + F_{cp} \cdot \frac{h}{T} \cdot \frac{a_2}{w}$$
$$F_{z,3} = \frac{1}{2} \cdot m \cdot g \cdot \frac{a_1}{w} - F_{cp} \cdot \frac{h}{T} \cdot \frac{a_1}{w}$$
$$F_{z,4} = \frac{1}{2} \cdot m \cdot g \cdot \frac{a_1}{w} + F_{cp} \cdot \frac{h}{T} \cdot \frac{a_1}{w}$$



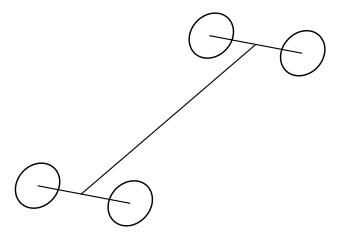
What does weight transfer depend on?

- CoG height
- CoG distance from front/rear axle
- weight of car
- amount and direction of acceleration
- wheelbase
- track

nothing else!

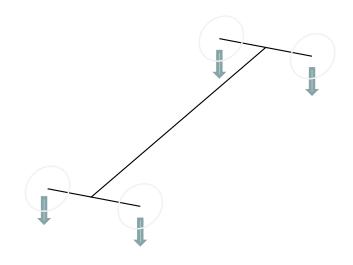
$$F_{z,4} = \frac{1}{2} \cdot m \cdot g \cdot \frac{a_1}{w} + F_{cp} \cdot \frac{h}{T} \cdot \frac{a_1}{w}$$
$$F_{cp} = m \cdot a_y$$

- CoG height
- CoG distance from front/rear axle
- weight of car
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- wheelbase
- track





- CoG height
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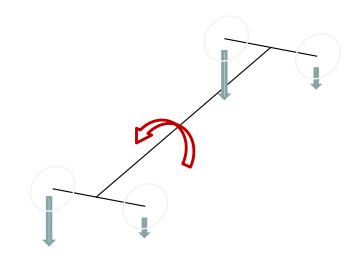


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AUTOMOTIVE TECHNOLOGIES

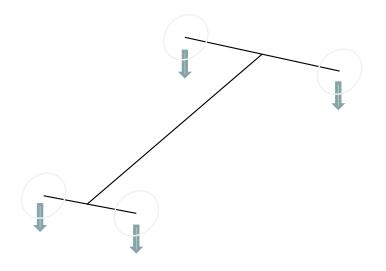
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- CoG height
- CoG distance from front/rear axle
- weight of car
- amount and direction of acceleration
- wheelbase
- track



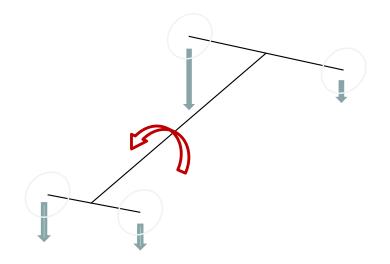


- CoG height
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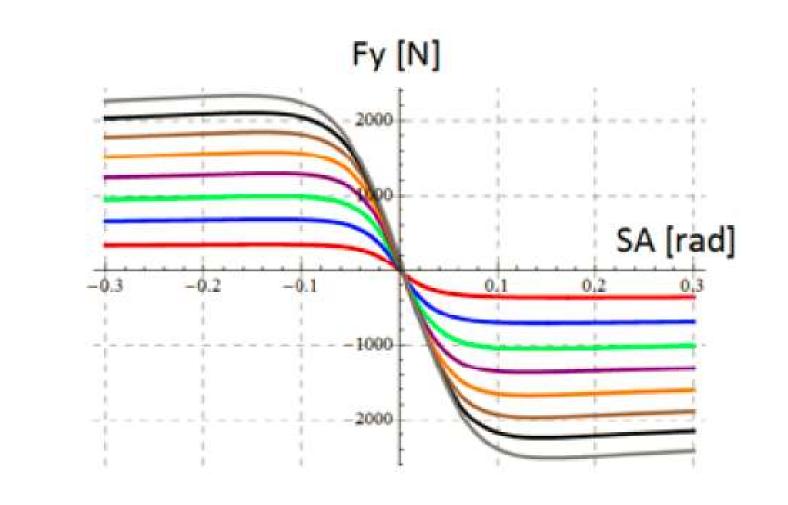
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...with tyre degressivity

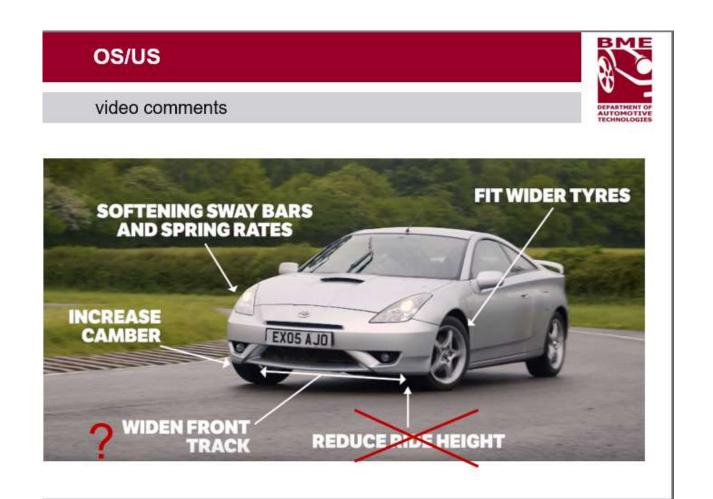




- Fz=200 - Fz=400 - Fz=600 - Fz=800 - Fz=1000 - Fz=1200 - Fz=1400 - Fz=1600

video comments









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1



- know concepts and definitions you are able to give definitions of :
 - different type of tyre radius
 - contact patch
 - tyre structures
 - slip ratio
 - slip angle
 - aware of the different characteristics of tyre behaviour and able to distinguish one from other
 - friction coefficient
 - brush tyre model and explanation of tyre force
 - able to orientate in the coordinate system of a vehicle
 - cornering stiffnes of a tyre
 - self aligning torque
 - pneumatic trail
 - friction ,cirle'
 - steady state basics equations
 - transient basics equation
 - characteristics of transient basics diagrams

2



- assymetric tyre behaviour to acceleration and braking
- static vertical tyre loads
- longitudinal weight transfer with the help of longitudinal model
- lateral weight transfer in steady state cornering
- understanding the effect of tyre degressivity and weight transfer

Bibliography



- https://www.google.com/url?sa=i&url=https%3A%2F%2Fwww.nhra.com%2Fnews%2F2019%2Fnhra-announces-2020-lucas-oildrag-racing-seriesschedule&psig=AOvVaw1gEiiWHmijOxHpbS0OT3sU&ust=1615131277553000&source=images&cd=vfe&ved=0CAIQjRxqFwoTCM DC4IT m-8CFQAAAAAdAAAABAD
- Optimum G Seminar by Claude Rouelle 2016 Graz
- https://www.google.com/url?sa=i&url=https%3A%2F%2Fwww.vectorstock.com%2Froyalty-free-vector%2Fcar-frontview-icon-image-vector-

12155010&psig=AOvVaw1hu57p4wtvijLerVJucABs&ust=1615139654671000&source=images&cd=vfe&ved=0CAIQjRxqFwoTCIiAq p-enO8CFQAAAAAdAAAAABAj

• https://tudasbazis.sulinet.hu/hu/szakkepzes/kozlekedes/kozlekedesi-alapismeretek/az-iv-sugara/a-kicsuszasi-es-a-kiborulasihatarsebesseg-ivmenetben-ii

Thank you for your attention!

