

Department of Automotive Technologies – Vehicle Mechanics Fundamentals

Gábor Sipos



Lecture 1

Introduction

contact: gabor.sipos.uni@gmail.com

Subject

Goal

Participation: Lectures and labs

Tests

Tests for semester signature ~6&13th weeks+Semester championship documentation

Based on the result of tests, final mark can be obtained, and no need for exam

Exam

Note

To prepare for exam/tests presentations will be uploaded to Moodle, but the some of the slides aren't – therefore to have the full picture lectures shall be attended.

- Milliken and Milliken, 1996 W.F. Milliken, D.L. Milliken Race Car Vehicle Dynamics, Society of Automotive Engineers Inc., Great Britain (1996)
- H. B. Pacejka 2006 Tyre and Vehicle Dynamics Butterworth-Heinemann
- Racecar: Searching for the Limit in Formula SAE Matt Brown Seven Car Publishing 2011

Basic informations

Week nr.	Date		Lecture (Monday)		Lab (date+1;Tuesday)
1	12th Feb	1	General information, Tyre, Driving force	1	Lab
2	19th Feb	2	Longitudinal and lateral behaviour		
3	26th Feb	3	Concepts and over/understeer	2	Lab
4	4th Mar	4	Weight transfer		
5	11th Mar	5	Bicycle model	3	Lab
6	18th Mar	6	Midterm exam I. ONLINE		
7	25th Mar	T1	Systems of the vehicle ONLINE	4	Lab ONLINE
8	1st Apr	-	Break		
9	8th Apr	7	Systems of the vehicle	T1 R	Exam 1 - subsequent
10	15th Apr	8	Quarter vehicle model ONLINE		
11	22th Apr		Break		Break
12	29th Apr	T2	Midterm exam II. ONLINE		
13	6th May	9	Tyre management	T2 R	Exam 2 - subsequent
14	13th May	10	Racecar engineering		

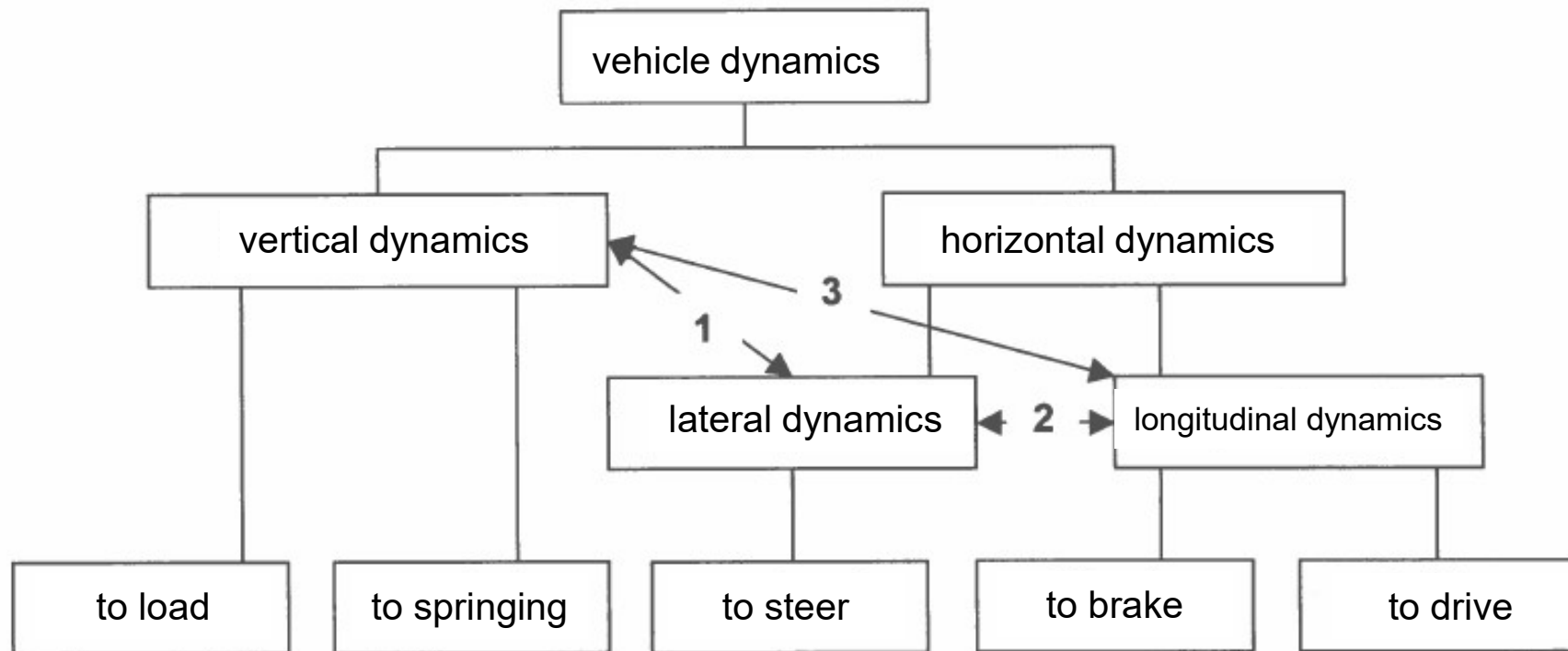
Some mechanical aspects

	Reason of the motion	Inertia	Equation of motion	Dynamics principle
Rectilinear motion	F [N] force	m [kg] mass	$F = m \cdot a$	$\dot{i} = \sum F$ i : Impulse derivative
Circular motion	M [Nm] torque	θ [kg m ²] Moment of inertia	$M = \theta \cdot \varepsilon$	$D_c = \sum M$ D _c : kinetic torque

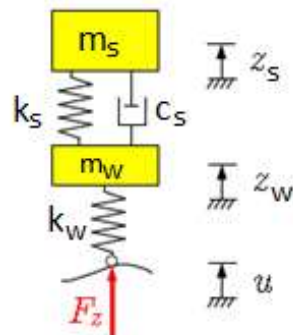
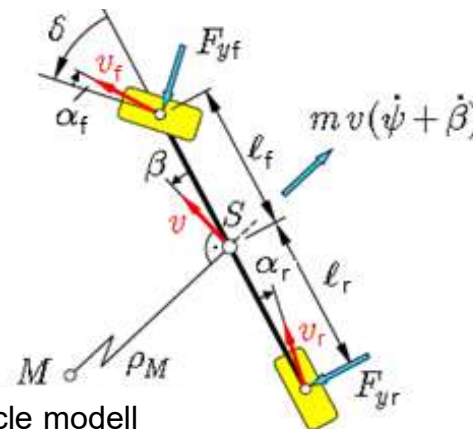
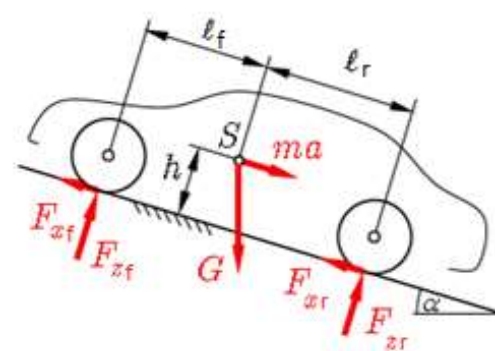
	Quantities		W Work done by	Work-energy principle
Rectilinear Motion	$I = m \cdot v$ impulse	$E = \frac{1}{2} \cdot m \cdot v^2$ Kinetic energy	$W = F \cdot \Delta s$	$W = \Delta E_{kinetic}$
Rotational motion	$N = \theta \cdot \omega$ Angular moment	$E = \frac{1}{2} \cdot \theta \cdot \omega^2$ Rotational energy	$W = M \cdot \Delta \varphi$	$W = \Delta E_{rotational}$

Some mechanical aspects

Vehicle dynamics blocks



Simple model of VD

	vertical dynamics	lateral dynamics	longitudinal dynamics
	 <p>Quarter vehicle modell</p>	 <p>Bicycle modell</p>	 <p>longitudinal modell</p>
Features	<ul style="list-style-type: none"> • Vertical vibration • Wheel loads • comfort 	<ul style="list-style-type: none"> • steering • Lateral acceleration • Self aligning torque • Critical speed 	<ul style="list-style-type: none"> • Acceleration and braking • resistances: tyre,air, uphill – power requirement; • Engine characteristics and gears • Braking and driving forces
Parameters	<p>m_s – structural weight m_w – wheel weight k_t – tyre stiffness k_s – spring stiffness c_s – damping u - movement</p>	<p>l – wheelbase l_f – CoG distance m – weight v – vehicle speed α – slip angle δ – steering angle</p>	<p>h – CoG height G – gravity force F_{xf} – front axle driving force F_{zf} – tyre forces at front</p>

Why racecars?

Abstract thinking

What's the main function of a racecar?



Why racecars?

Interview question of McLaren F1

Why porpoising affects more the top teams?



F1 2022 EARLY TALKING POINTS



Why racecars?

Interview question of McLaren F1

Why porpoising affects more the top teams?
If you are power limited rather than grip-limited, you do not really lose any performance. It mostly happens in high speed corners and straights.

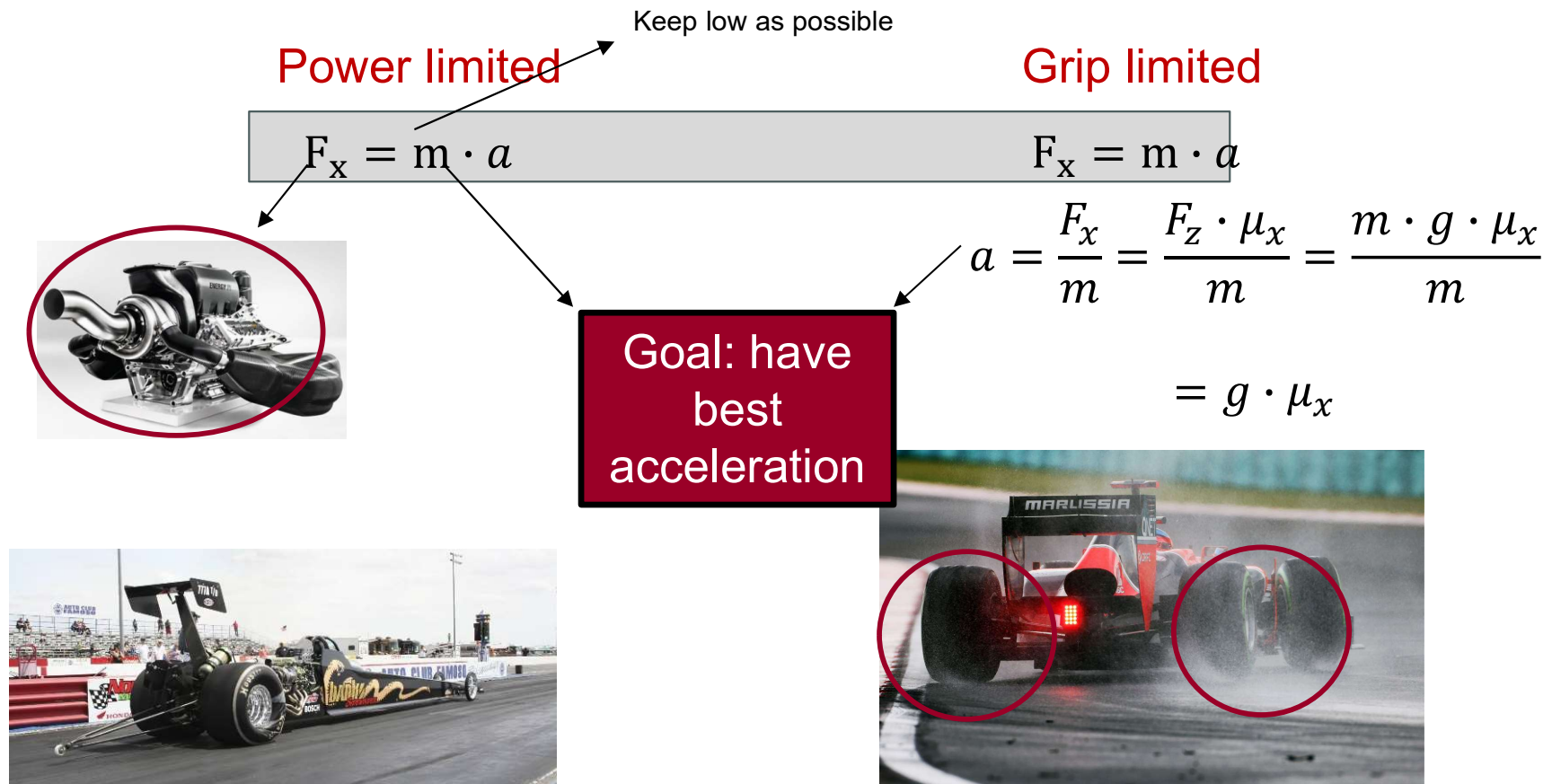


F1 2022 EARLY TALKING POINTS



Grip or power limited?

Newton's second law



Why racecars?

BMW advertisement

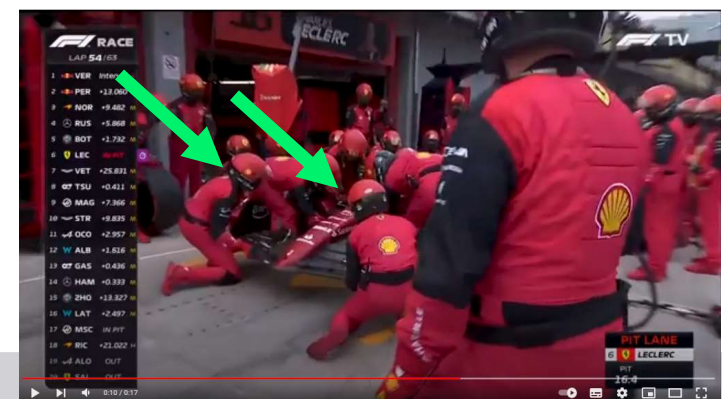
Good handling is possible only with perfect weight distribution?

No, WD is one element, it must match with cornering stiffness balance to reach good handling.



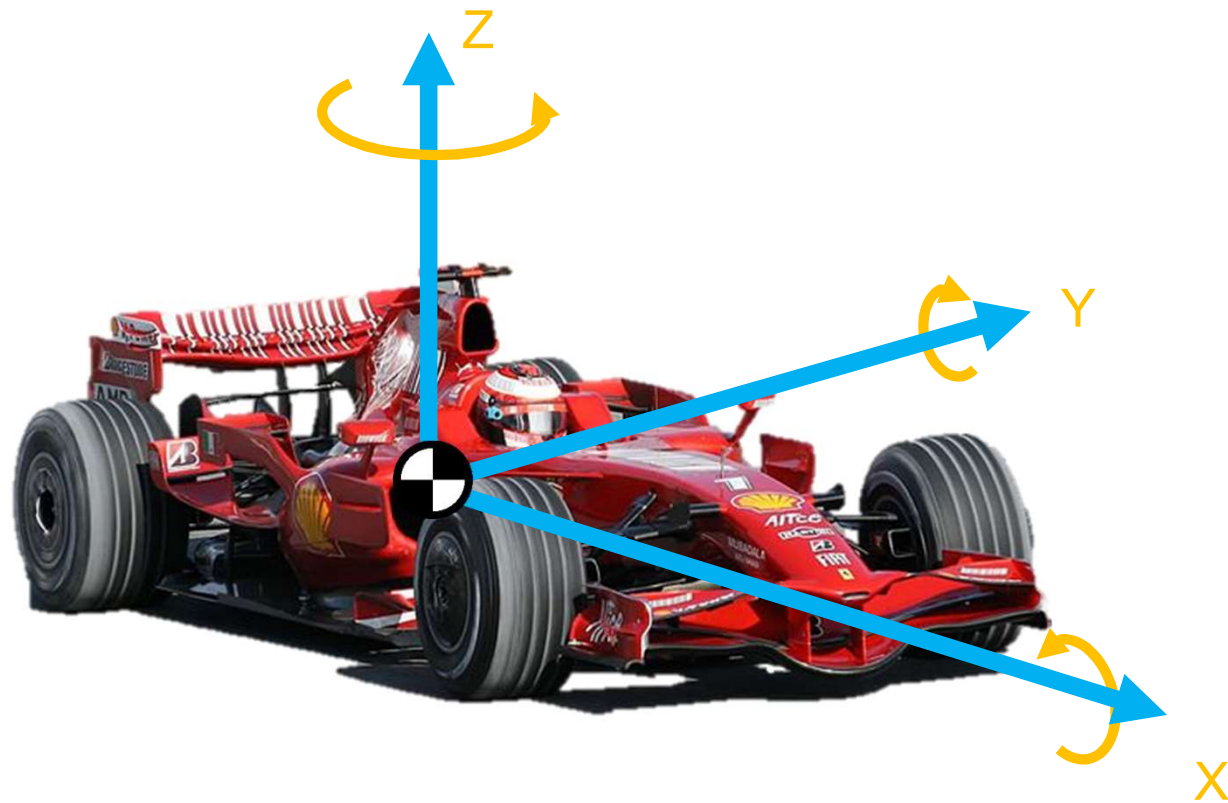
Why racecars?

Front wing change F1

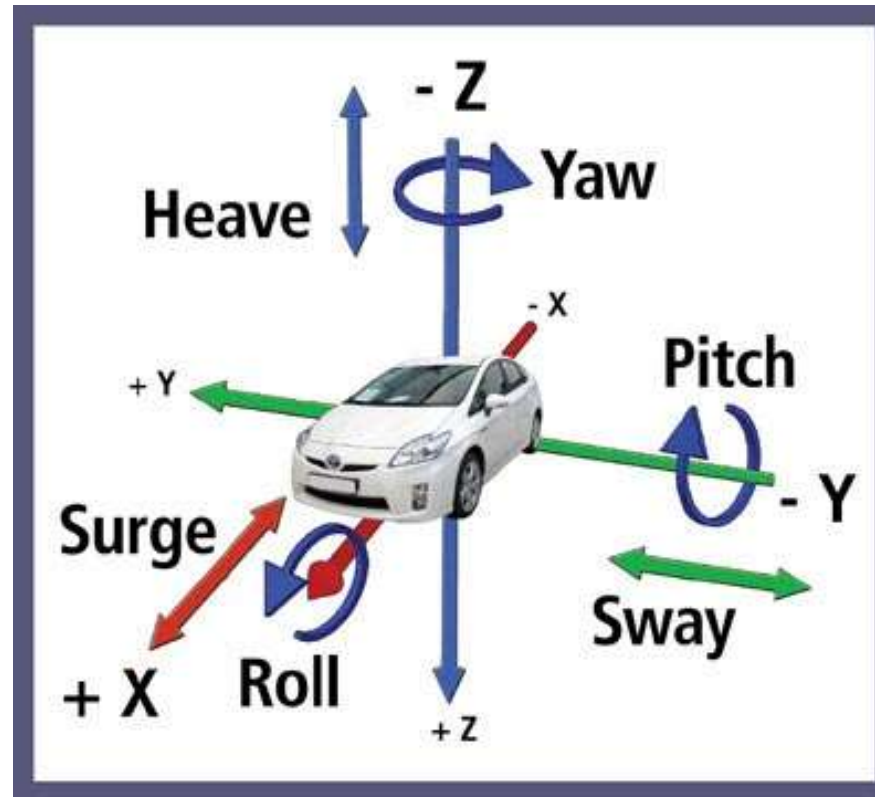


Coordinates in vehicle dynamics

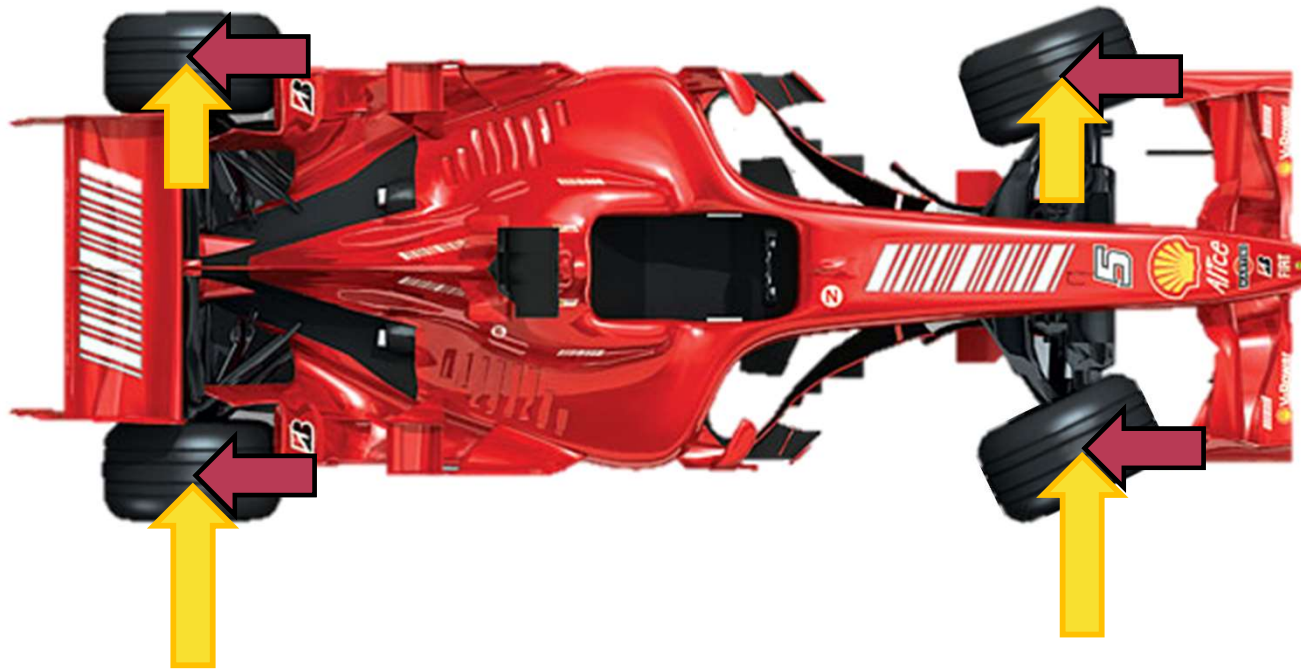
DoF



Coordinates in vehicle dynamics



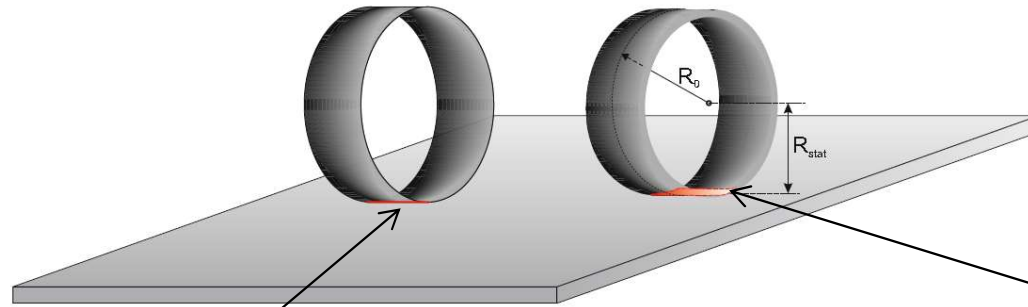
Vehicle dynamics fundamentals



- Tyre
 - the only contact with the ground - ideally
 - **The most important part of the car.**
- Racing tyre
 - The most important part of the car.
 - The suspension, the aerodynamics, the powertrain are areas that try to optimize the tyre performance
 - The main feedback to the driver comes from the tyre
 - Why racecars?

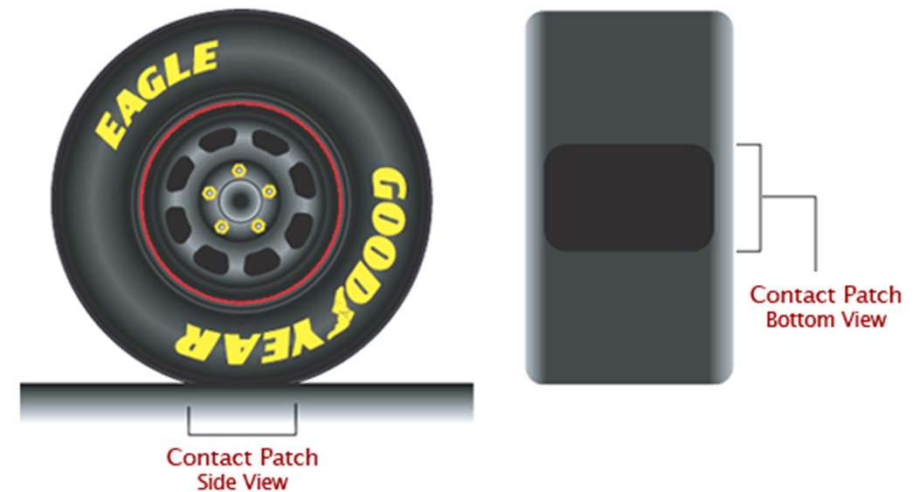


Contact with the ground

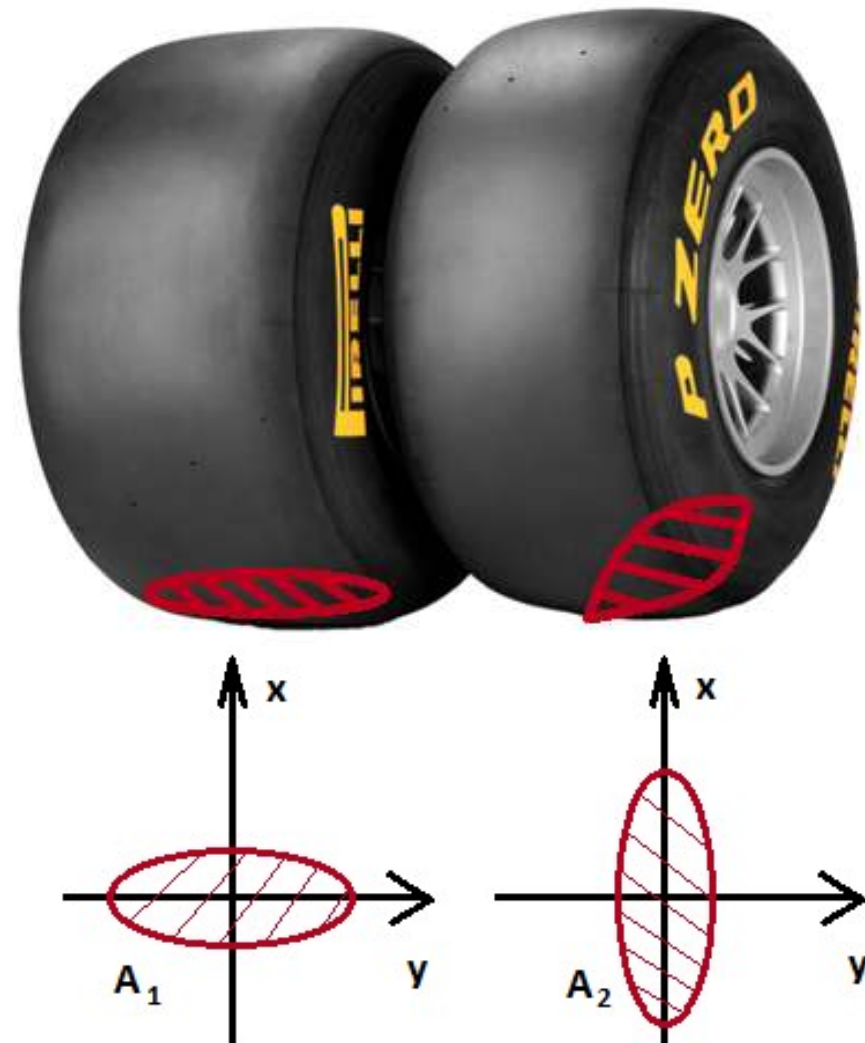


rigid cylinder, linear contacting

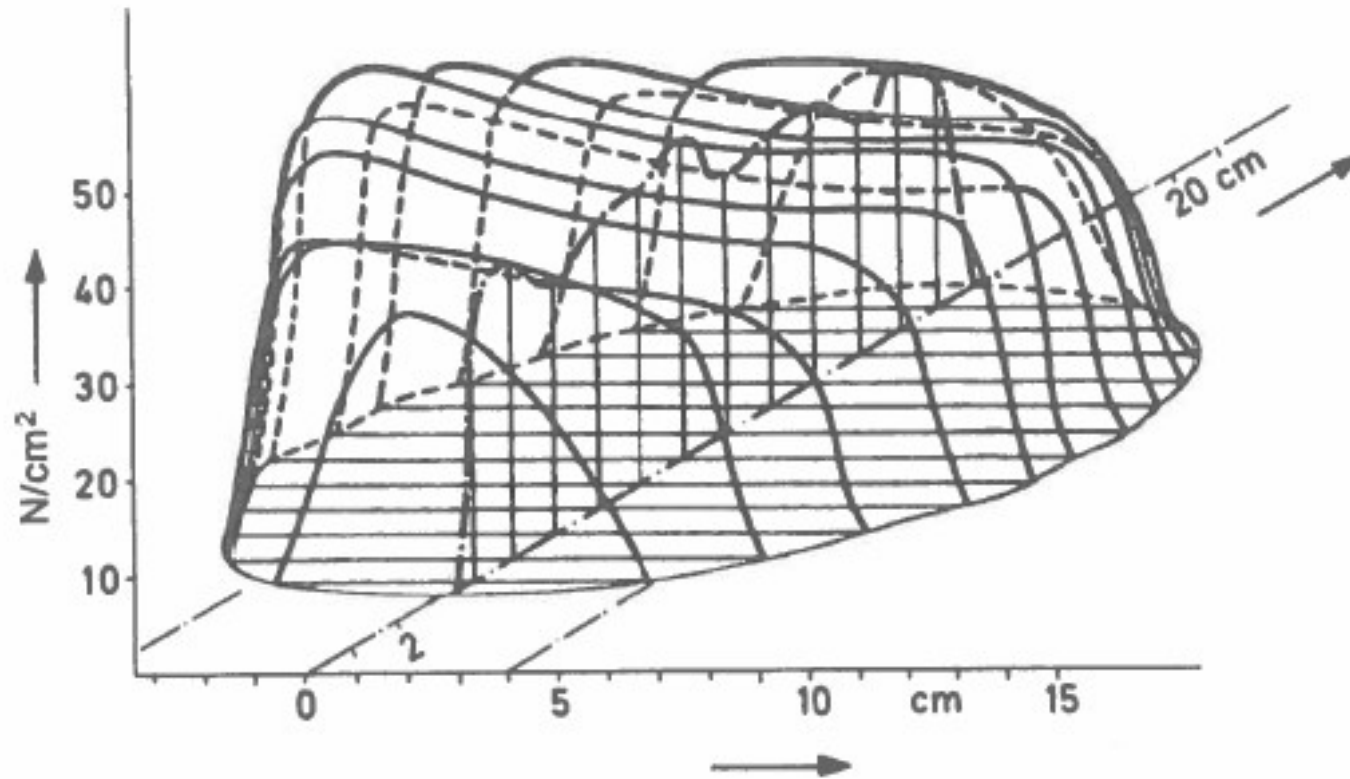
elastic tyre,
contact patch (postcard-sized)



Contact with the ground

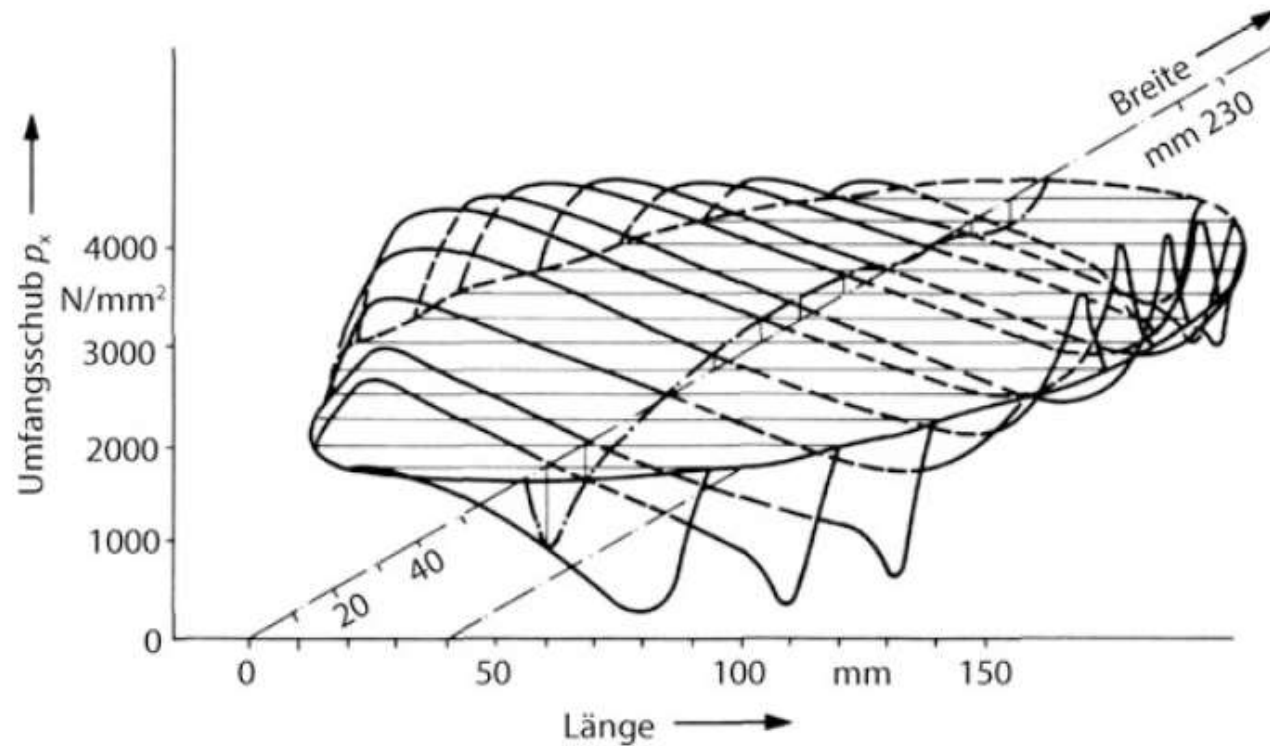


Contact with the ground



Load distribution along the contact patch – not driven wheel

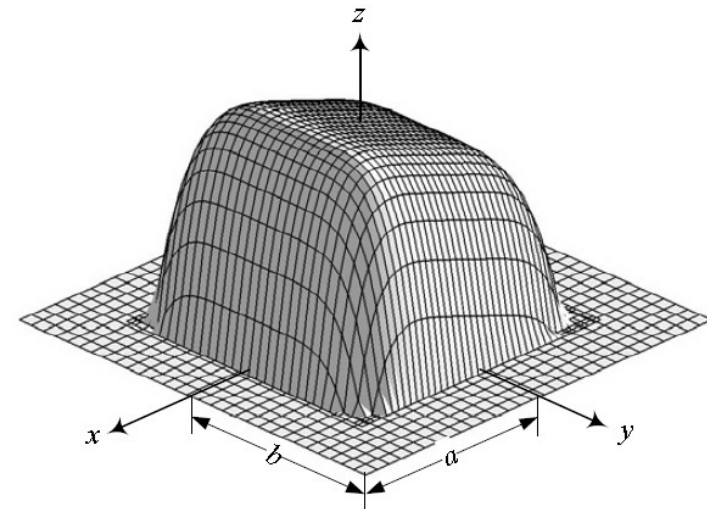
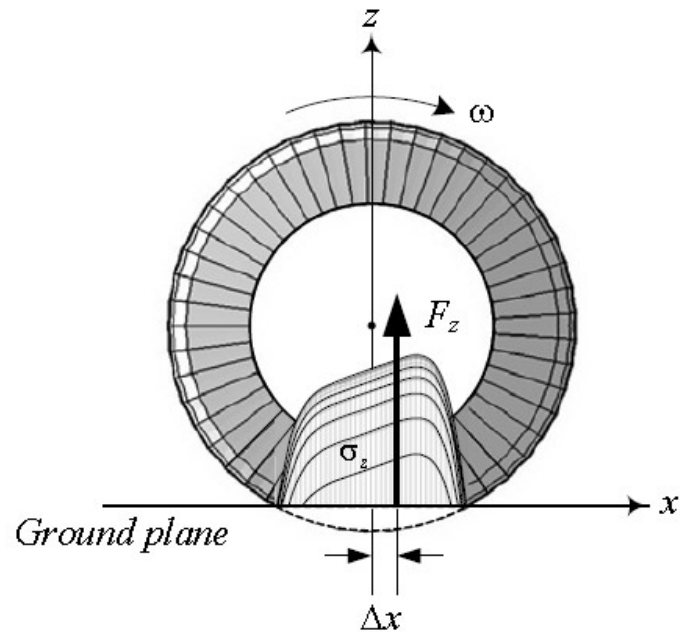
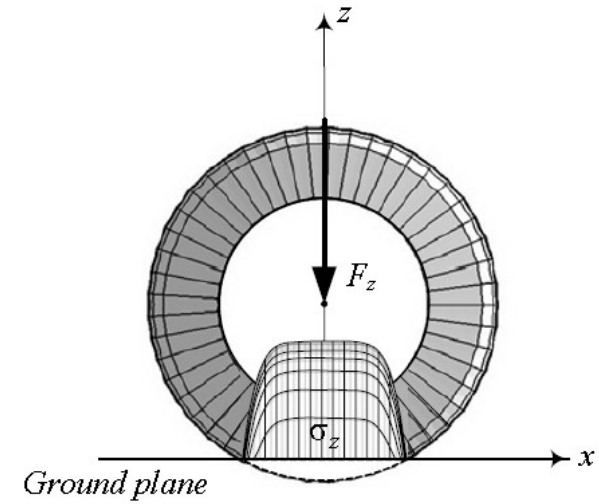
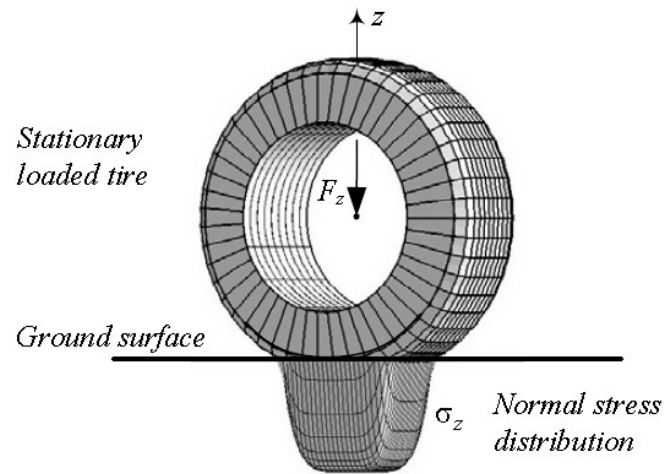
Tangential stress



Load distribution along the contact patch – not driven wheel

Source: Reimpell, J., Sponagel, P.: Fahrwerktechnik: Reifen und Räder. Vogel-Verlag, Würzburg (1995)

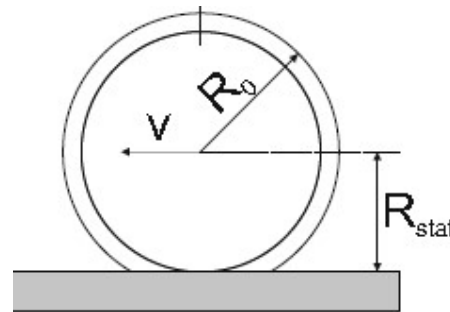
Contact with the ground



Static radius, R0

R_{stat} – Static radius is the distance measuring from center of the wheel to the supporting surface.

R_0 – Radius of a tyre unloaded, new condition.



Example: FS racecar with different R_0 radiuses caused top speed difference. What are the other effect(s)?

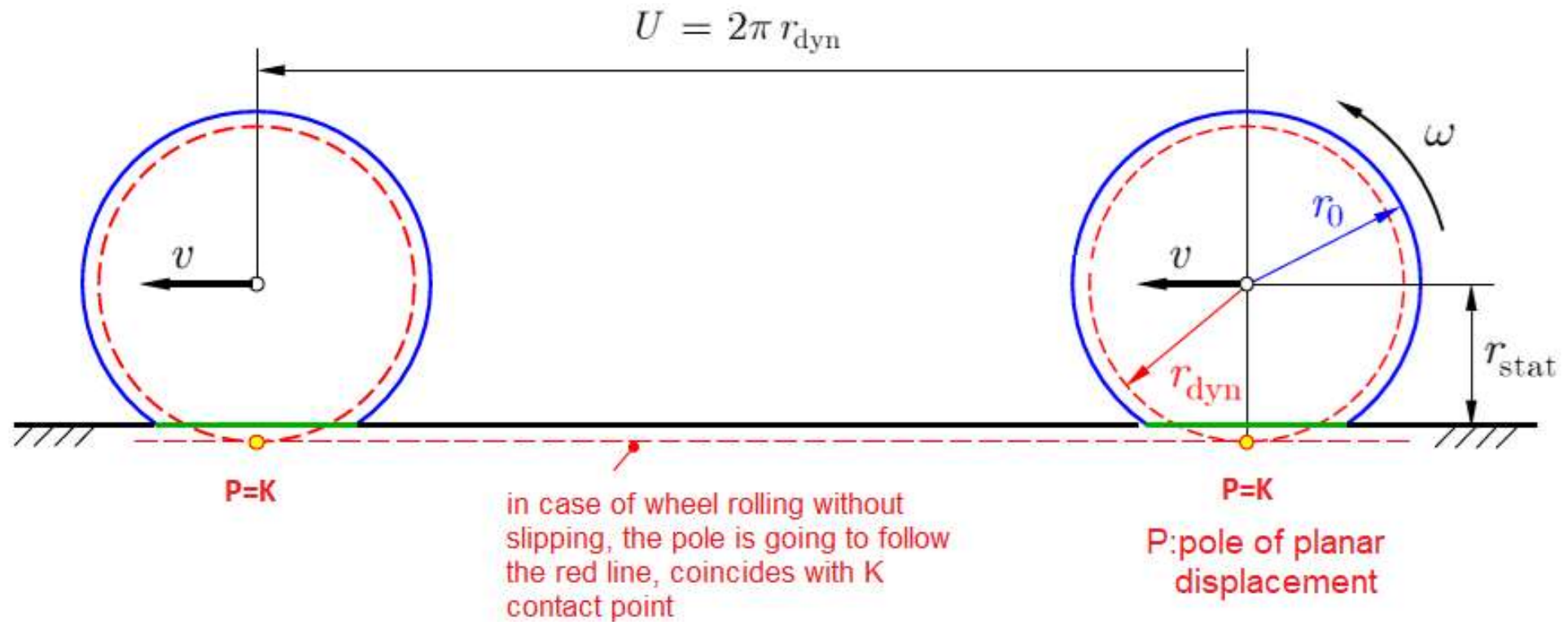
$$\vec{M} = \vec{r} \times \vec{F}$$

Contact with the ground

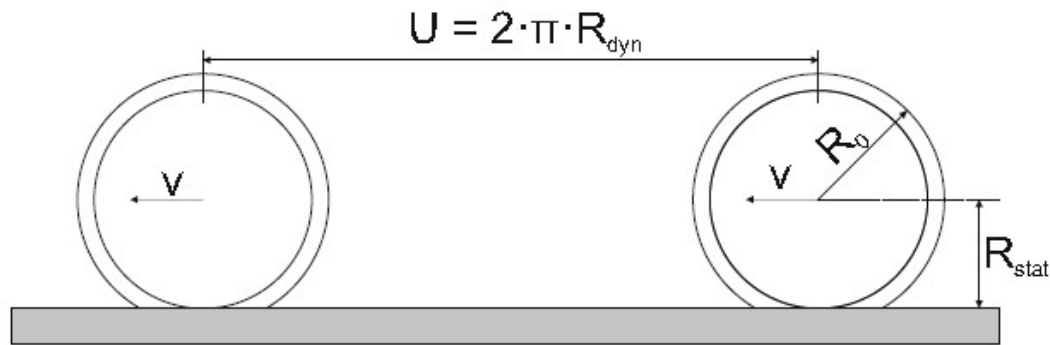
$$\vec{M} = \vec{r} \times \vec{F}$$



Dynamic radius



(source: C. Woernle)



Beispiel:

Reifen	r_0 [mm]	r_{stat} [mm]	U [mm]	r_{dyn}
185/65 R 15	311	284	1895	302
195/65 R 15	318	290	1935	308
205/65 R 15	324	294	1975	314

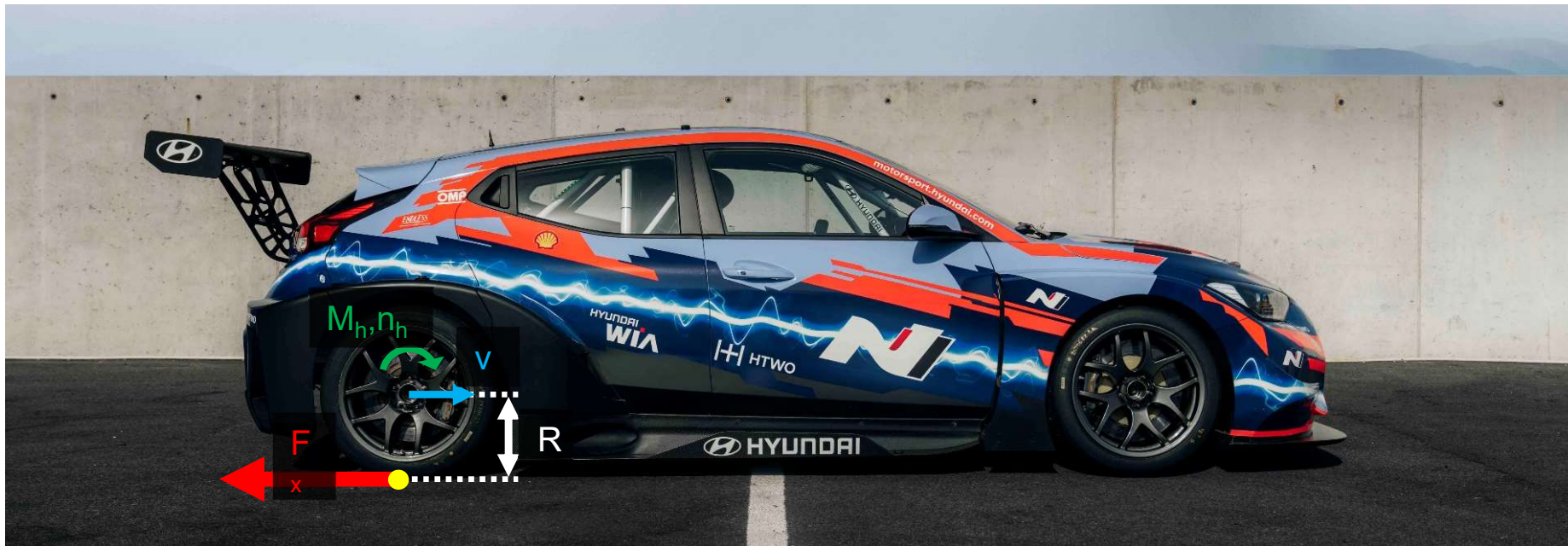
Quelle: Reimpell 1995

- The typical value of R_{dyn} is between R_0 and R_{stat} .
- If we mark as f bumping the difference between R_0 and R_{stat} , then we can say $R_{dyn} = R_{stat} + 2/3f$

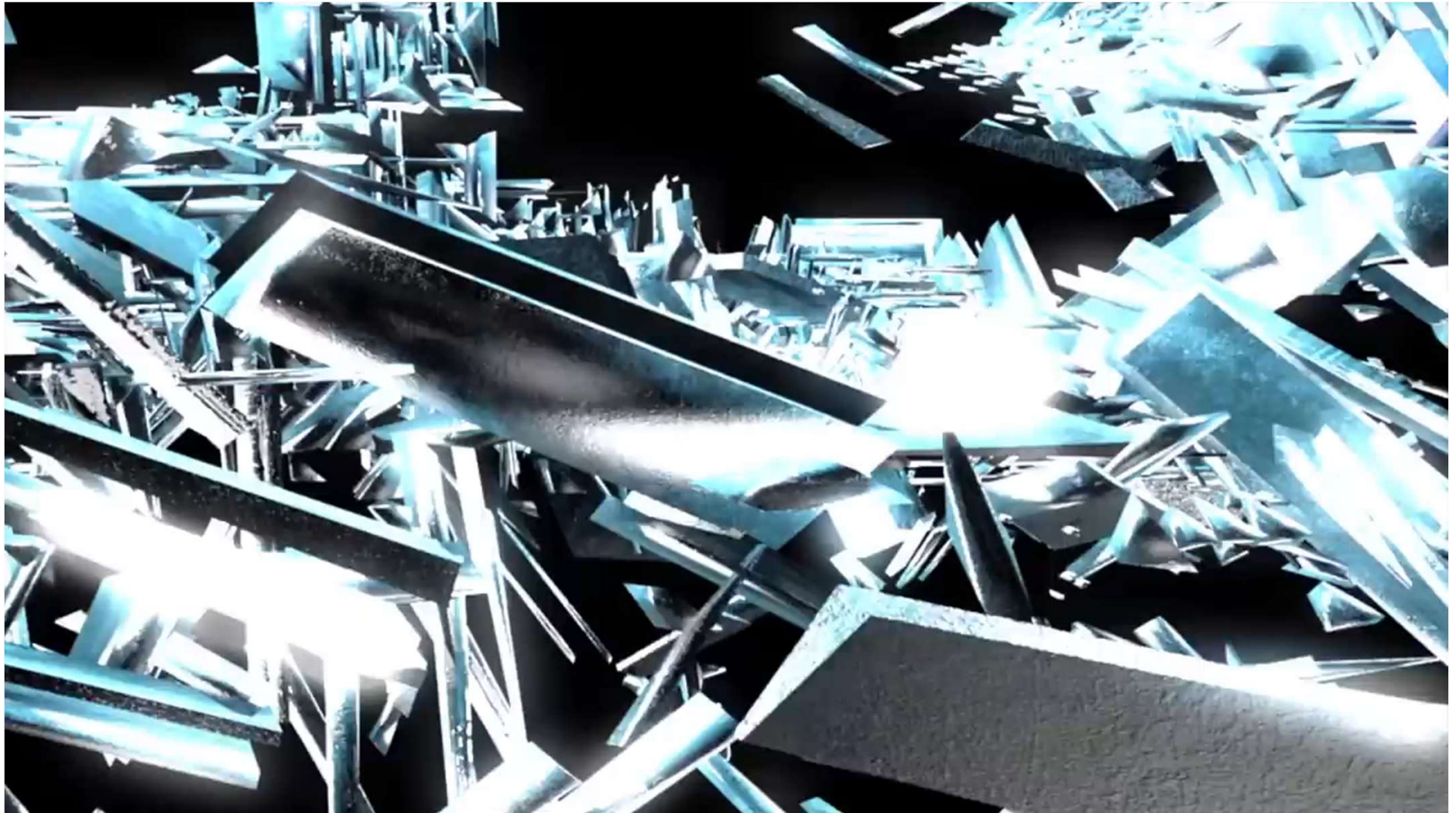
Dynamic radius

$$M_h = R \cdot F_x$$

$$v = R \cdot \omega = R \cdot 2\pi \cdot n_h$$



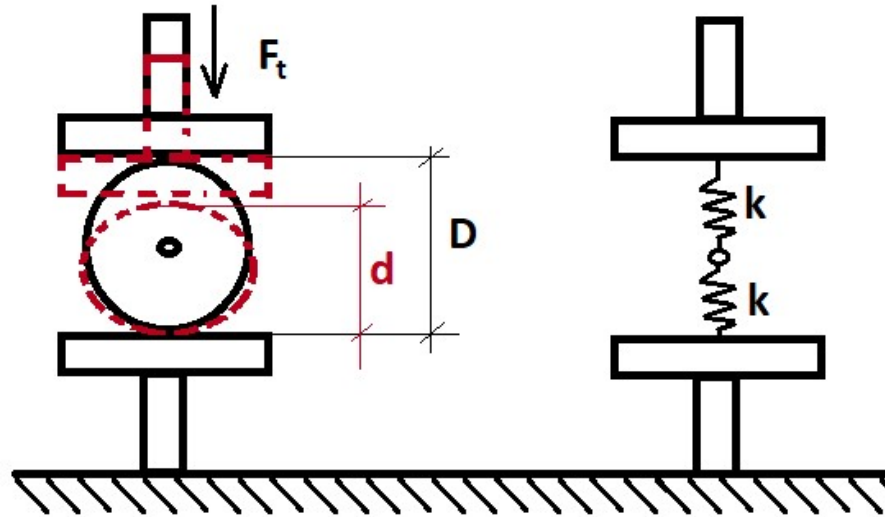
Different radiuses



Tyre vertical stiffness



Tyre vertical stiffness

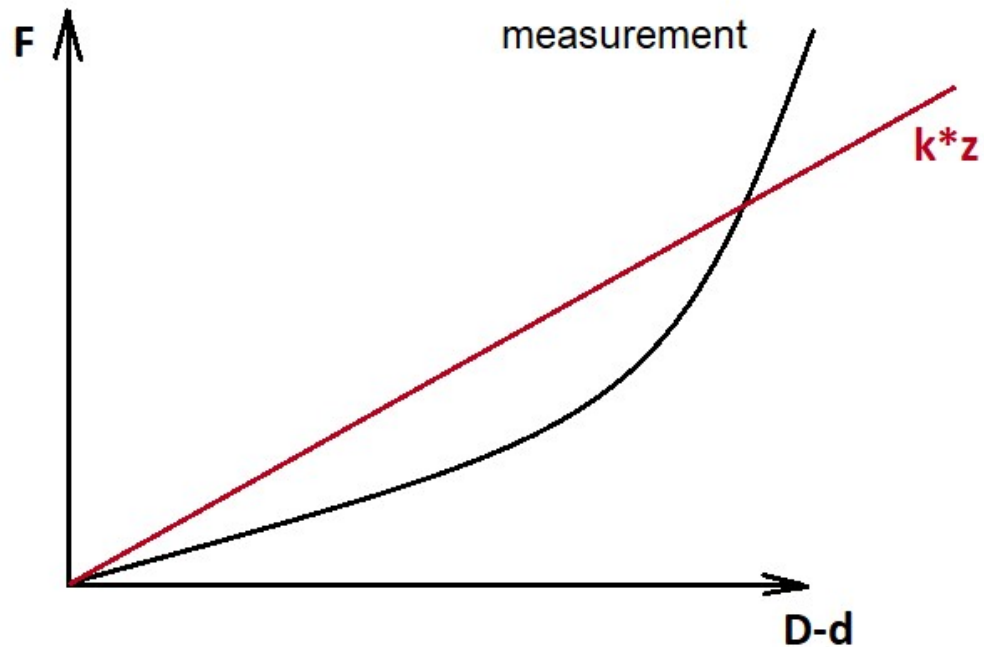


$$k = \frac{F_t}{\left(\frac{D-d}{2}\right)} \left[\frac{N}{m}\right]$$

$$F_z = \frac{m \cdot g \cdot \%_F}{2} \text{ [N]}$$

$$R_{stat} = R_0 - \frac{F_z}{k} \text{ [mm]}$$

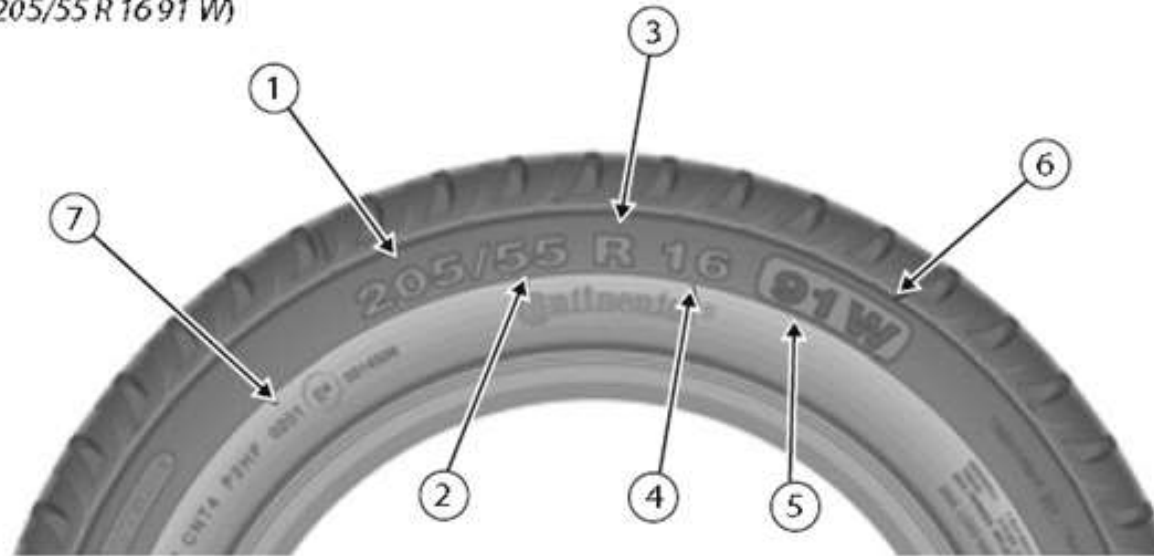
- k: spring stiffness; F_z : normal force ; $\%_F$: first axle load; m: vehicle weight



- Progressivity
 - effect of air
 - tyre effect
 - sidewall structure
 - sidewall size
- Curve fitting
 - approximation of stiffness
 - $k*z$

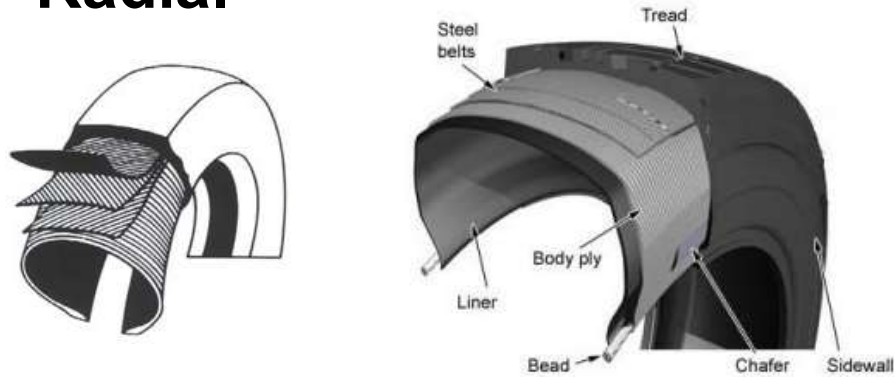
Tyre nomenclature

(z. B. 205/55 R 16 91 W)



- 205 – tyre width [mm]
- 55 – ratio between tyre height and width [%]
- R – structure (R/D)
- 16 – rim diameter – inches
- 91 – load capacity index
- W – speed limit (W=270 km/h)

Radial



Bias / Crossply construction

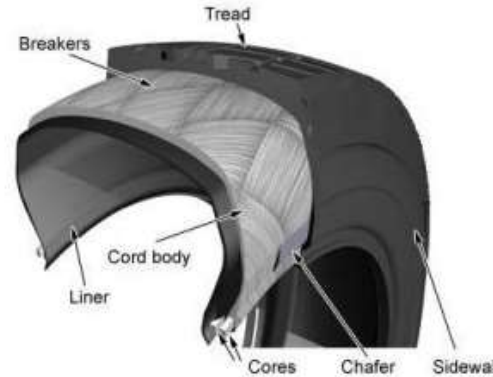


Image source: recstuff.com/radialvsbiasplytrailertires.aspx

Diagonal



- **Radial strings +belts**

- **Advantage**

- better roadability
- less sensitivity to pot-holes

- **Disadvantage**

- bigger driving noise
- higher costs

- **Crossplies**

- **Advantages**

- less vulnerable
- easier to manufacture

- **Disadvantages**

- lower spring-comfort
- poor manoeuvrability
- worse grip-warming-tyre wearing

Longitudinal slip – slip ratio

$$SR = \kappa = \frac{R_{dyn} \cdot \omega - v}{v} [-]$$

$$\kappa \in [-1; \infty]$$



braking



0

rolling

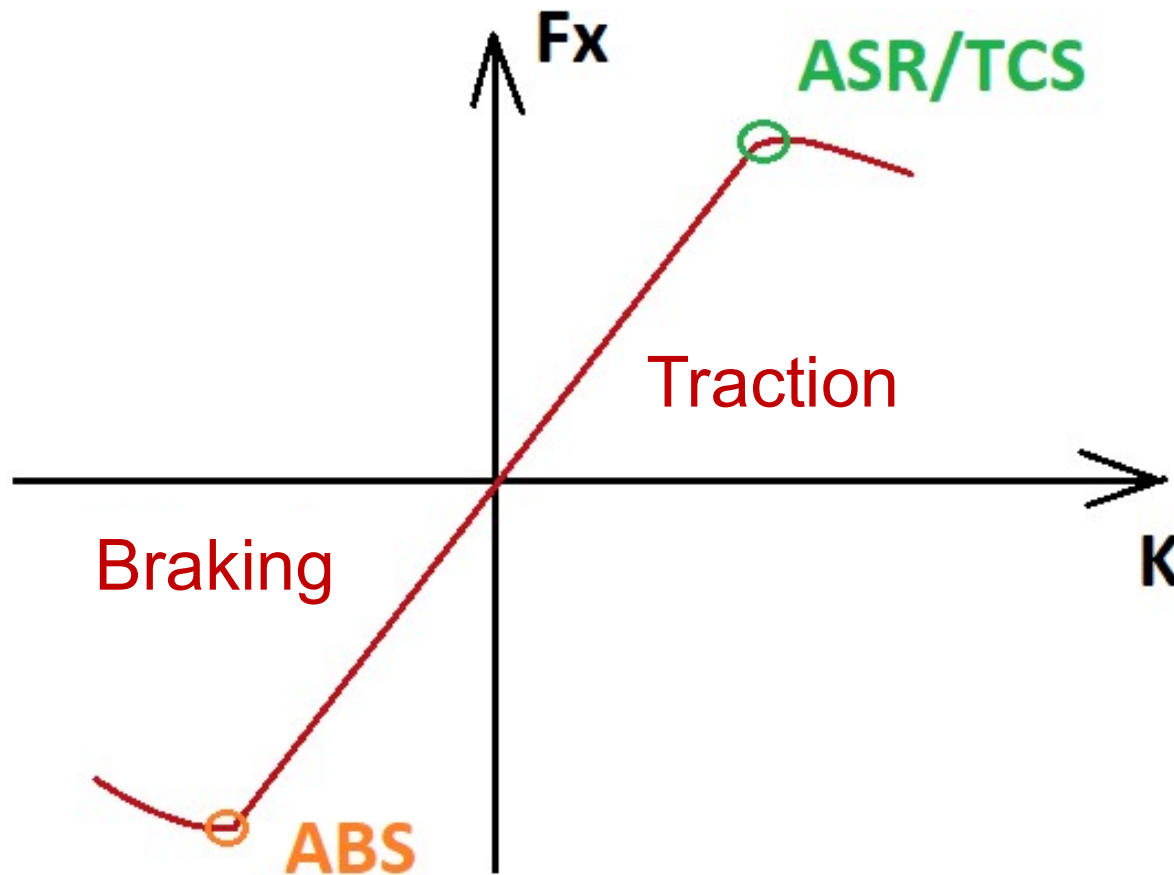


drive

v – vehicle speed [m/s]; ω – angular velocity [1/s]

Driving force

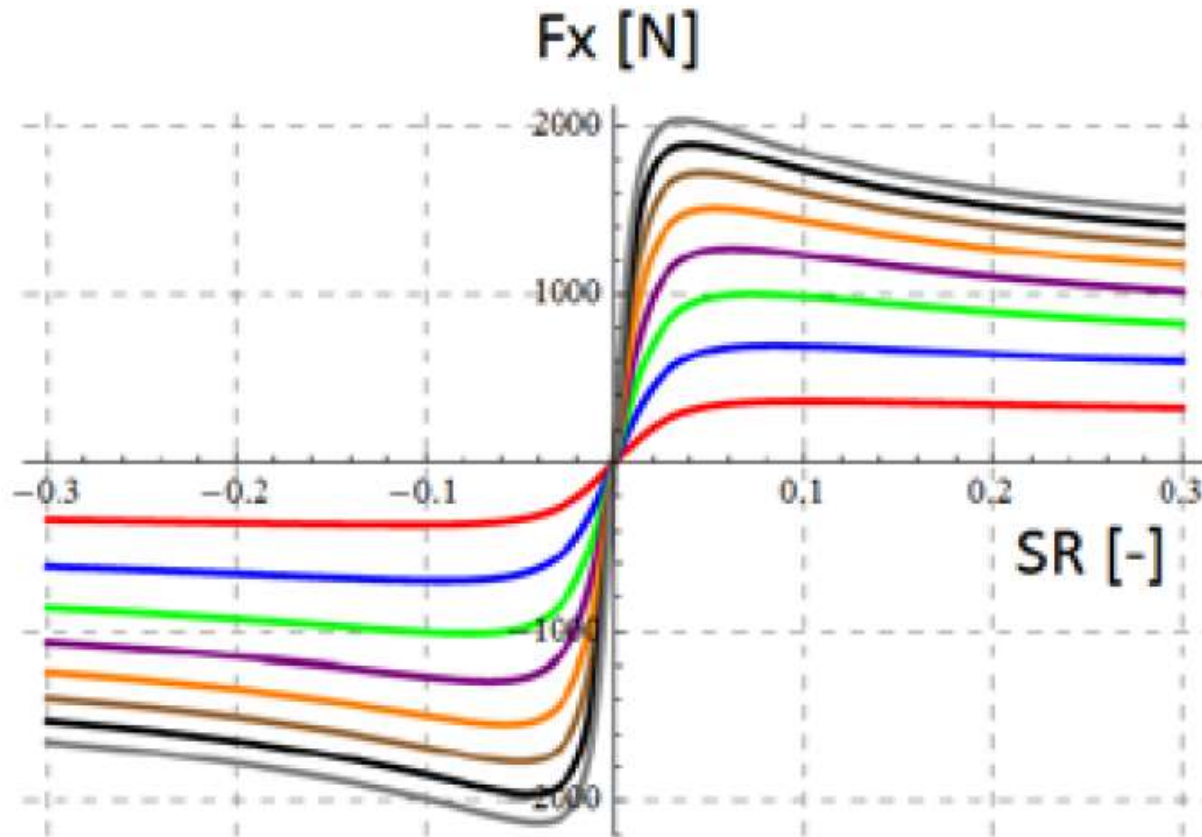
Characteristics depends on tyre!



$$SR = K = \frac{R_{dyn} \cdot \omega - v}{v} [-]$$

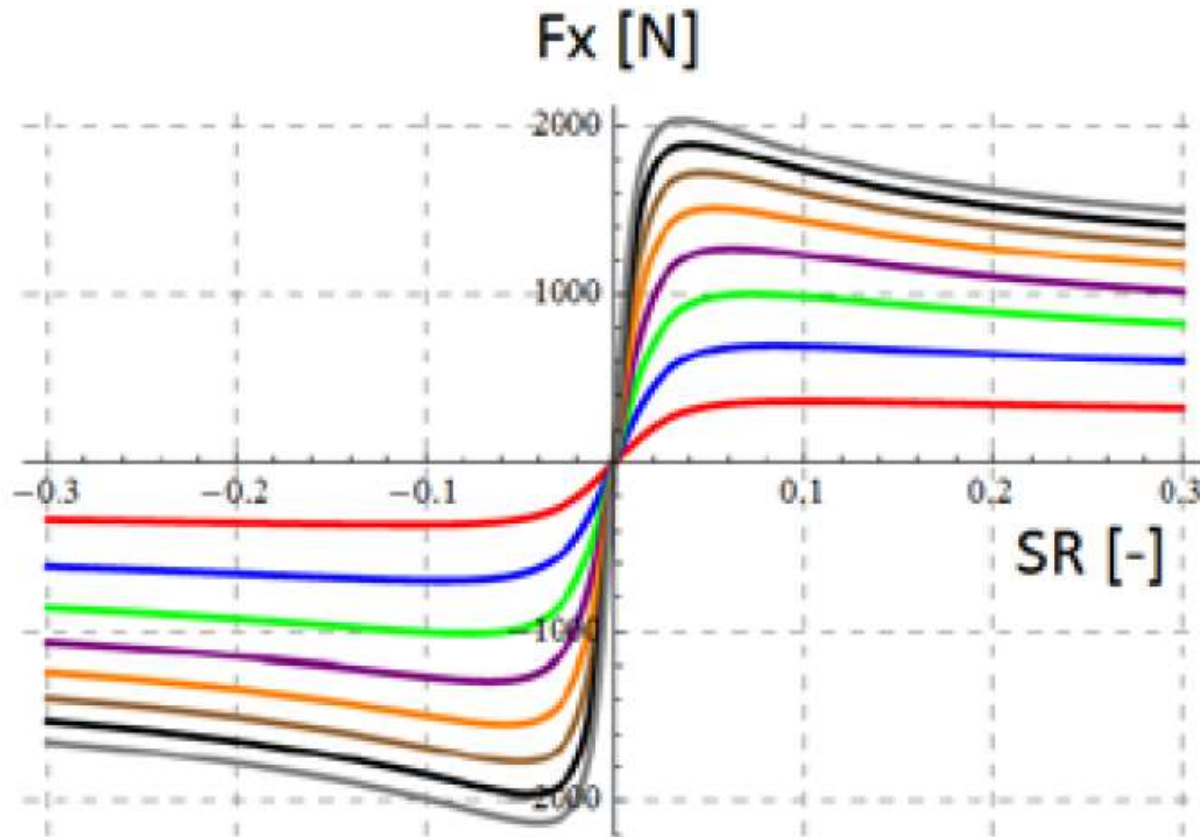
Driving force

What could the colours mean?



Driving force

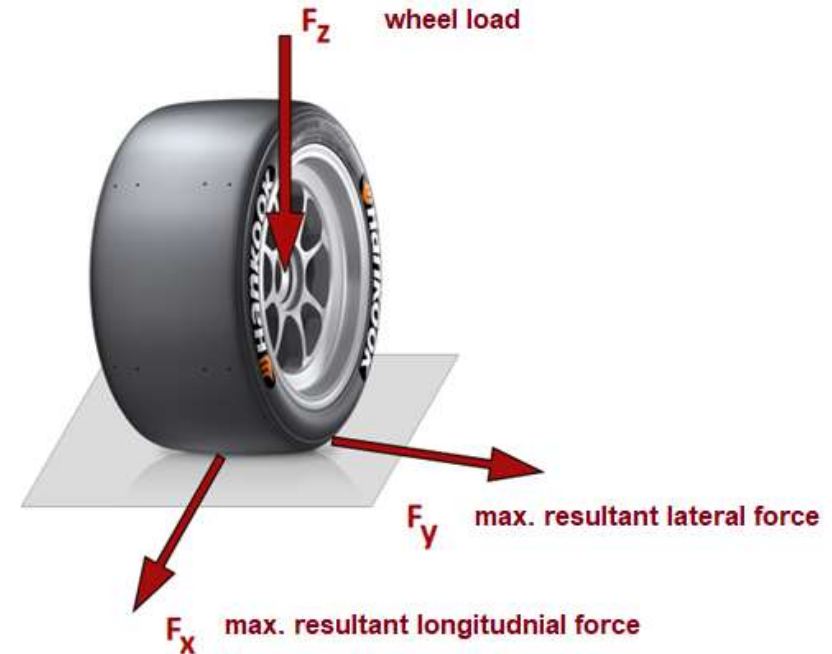
What could the colours mean?



— $F_z=200$ — $F_z=400$ — $F_z=600$ — $F_z=800$ — $F_z=1000$ — $F_z=1200$ — $F_z=1400$ — $F_z=1600$

Longitudinal friction coefficient

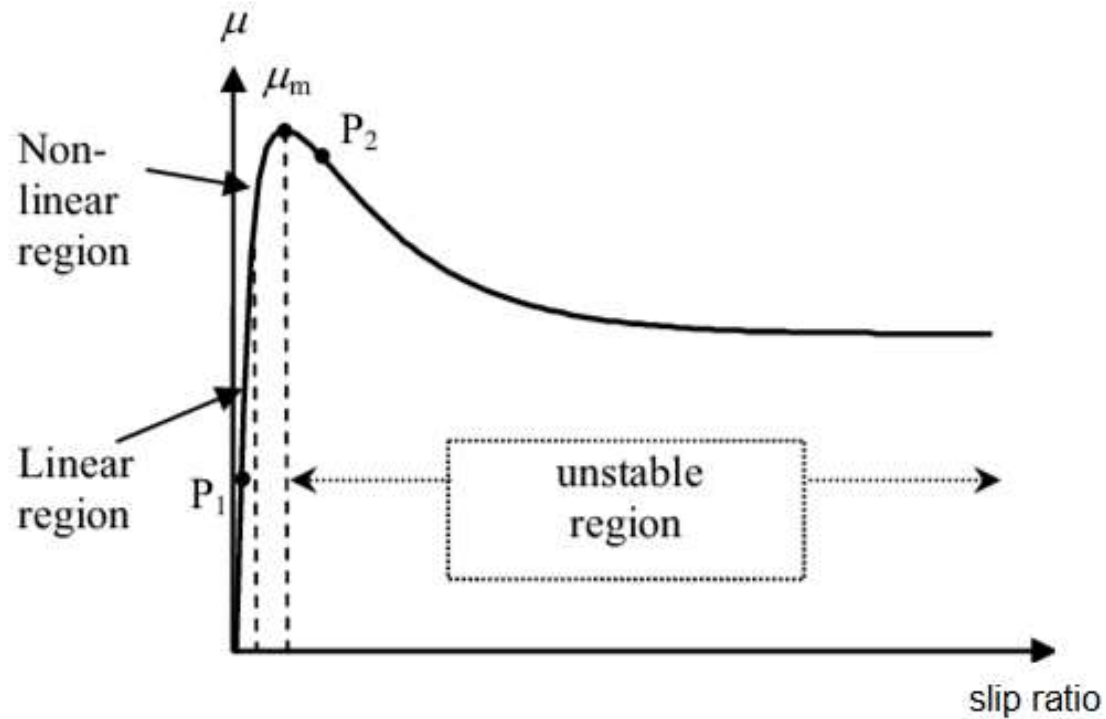
$$\mu_x = \frac{F_x}{F_z}$$



μ_x :

- longitudinal friction coefficient
- friction coef.

Typical μ_x -slip ratio curve

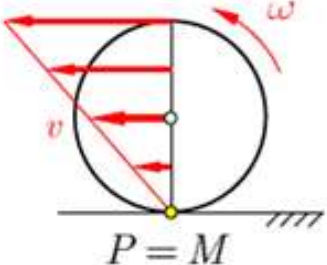
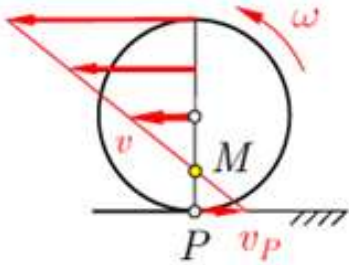
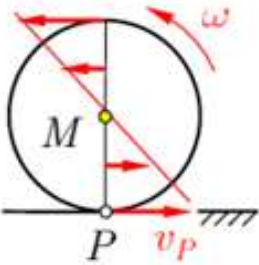
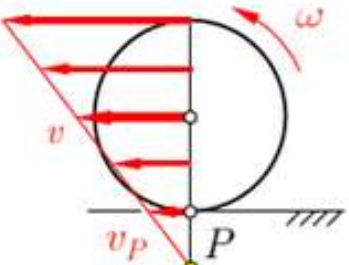
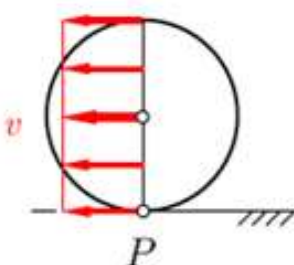


F_x belongs to maximum grip

$$F_{x,max} = \mu_{max} \cdot F_z$$

Driving force

Phases of slip ratio

$v = \omega R_{dyn}$	$v < \omega R_{dyn}$		$v > \omega R_{dyn}$	
rolling wheel	driven wheel	spinning wheel	braking	wheel blocked
 <p>$P = M$</p> <p>kontaktpont : P sebességpólus : M</p>	 <p>P v_p M</p>	 <p>M P v_p</p>	 <p>v_p P M</p>	 <p>P</p>
no slip	driving slip		braking slip	
$K = 0$	$K = \frac{R_{dyn} \cdot \omega - v}{v}$	$K = \infty$	$K = \frac{R_{dyn} \cdot \omega - v}{v}$	$K = -1$

Quelle: Popp, Schiehlen 1993

Tab. 3.6: Gegenüberstellung unterschiedlicher Schlupfdefinitionen am Beispiel des Längsschlupfes für ein in der Vertikalebene rollendes Rad

Antreiben Schlupf für Bremsen	$\frac{v_C - \omega r_e}{v_C}$	$\frac{\omega r_e - v_C}{v_C}$	$\frac{\omega r_e - v_C}{\omega r_e}$	$\frac{\omega r_e - v_C}{\omega r_e}$ $\frac{v_C - \omega r_e}{v_C}$
Quelle	wir	[3.18]	[3.19]	[3.20]
Schlupfwert für durchdrehendes Rad $\omega \rightarrow \infty$	$-\infty$	∞	1	1
Schlupfwert für blockiertes Rad $\omega = 0$	1	-1	$-\infty$	1
Schlupfwert für frei rollendes Rad $v_C = \omega r_e$	0	0	0	0

v_C Translationsgeschwindigkeit des Radschwerpunktes in Richtung der Radebene
 ω Winkelgeschwindigkeit des Rades um die Radachse
 r_e effektiver Rollradius

- https://www.google.com/url?sa=i&url=https%3A%2F%2Fwww.wired.com%2F2008%2F05%2Fff-formulaone%2F&psig=AOvVaw1Jpc66dH0FE0zK_Le6ev-D&ust=1612202433180000&source=images&cd=vfe&ved=0CAIQjRxqFwoTCNDU7Z_gxu4CFQAAAAAdAAAAABAD
- <https://www.google.com/url?sa=i&url=https%3A%2F%2Fthecuriousastronomer.wordpress.com%2Ftag%2Fangular-velocity%2F&psig=AOvVaw0VWlzbG2-A6w7ca5mULVB&ust=1612208234344000&source=images&cd=vfe&ved=0CAIQjRxqFwoTCJjTkuv1xu4CFQAAAAAdAAAAABAY>
- https://www.google.com/imgres?imgurl=https%3A%2F%2Fi0.wp.com%2Fotrwheel.com%2Fwp-content%2Fuploads%2F2016%2F08%2FradialvsbiasplyFINAL.jpg%3Fresize%3D630%252C301&imgrefurl=https%3A%2F%2Fotrwheel.com%2Fotr-blog%2Fradial-vs-bias-need-know%2F&tbnid=y0Ea_MrsrdG3gM&vet=12ahUKEwi4sPLjvM7uAhUjqnEKHTx0AewQMygFegUIARCiAQ..i&docid=0Oz8itZjkydoIM&w=630&h=301&q=tyre%20%20bias%20ply%20radial&ved=2ahUKEwi4sPLjvM7uAhUjqnEKHTx0AewQMygFegUIARCiAQ

Thank you for your attention!

